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SPDE/SPRE Final Summary Report

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1.0 INTRODUCTION

To support the objectives of the SP-100 program in assessing the applicability of power conversion technologies to a space-based power generating system, NASA-Lewis Research Center (NASA-LeRC) awarded an initial 17-month contract, NAS3-23883, to Mechanical Technology Incorporated (MTI) to design, fabricate, and test a space power demonstrator engine (SPDE). The goal of the SPDE program was to demonstrate the feasibility of free-piston Stirling engine (FPSE) power converter systems for space applications. The FPSE power converter offers the potential for extremely long life, high reliability, and excellent efficiency at low hot-to-cold temperature ratios, and can provide this efficiency at a relatively low heater head temperature. All of these attributes are attractive to a space power conversion system.

MTI's SPDE design consisted of two identical submodules in an opposed configuration. The SPDE was on test in 16 months and was operated at full design conditions. The engine was developed over the next year to achieve the majority of its design goals. When the engine development was complete, MTI and NASA-LeRC decided to separate the submodules of the engine to accelerate continued development of space power converter technology. The SPDE submodules were named space power research engines (SPREs), with SPRE-I located at NASA and SPRE-II located at MTI.

The purpose of this report is to summarize MTI's test and development activities within the SPDE/SPRE program. Section 2.0 presents an overview of the SPDE portion of the program. Section 3.0 describes results of SPRE testing, and Section 4.0 presents results of testing performed on SPRE power piston hydrodynamic bearings.

Initial SPDE tests indicated that the linear alternator efficiency shortfall of 70% (versus 90% design) was attributed to the magnetic structure surrounding the alternator. The SPRE alternator was tested on a linear dynamometer with both magnetic and nonmagnetic structures. Section 5.0 contains the results of this testing.

Finally, Section 6.0 describes MTI's design of a heat pipe heater head that would integrate with the SPRE and is a potential approach for a heat input system in advanced engines. This heat pipe design provides a design against which alternative designs can be measured.

2.0 SPDE OVERVIEW

The objective of the SPDE program was to design, build, and demonstrate with full-scale hardware the key technology issues that would permit selection of the FPSE generator as the space power conversion system. Key technology issues to be demonstrated included:

- 25-kW power output
- 25% system efficiency (electric power out/heat into the head)
- 8 kg/kW (17.6 lb/kW) specific power
- · Temperature ratio of 2.0
- Hot-end temperature of 630 K, cold-end temperature of 315 K
- · Successful application of hydrostatic internal gas bearings
- Dynamic balance (less than 0.08 mm) vibration amplitude along any axis.

2.1 SPDE Design

The SPDE consisted of two identical 12.5-kW_e FPSE submodules (shown in Figure 1) in an opposed (heater head to heater head) in-line configuration. Each submodule contains a linear, monocoil permanent-magnet alternator. The submodules are standard, spring-to-ground, virtual-rod-displacer engines with helium as the working fluid. The engine heater and cooler are annular, tube-in-shell units, with the engine working fluid passing through the tubes. The regenerator is an annular, stacked screen matrix sandwiched between the heater and cooler.

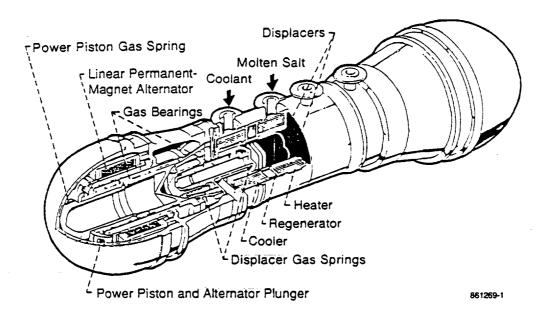


Figure 1. Space Power Demonstrator Engine

The permanent-magnet alternator is a moving magnet design in which the magnets are carried on a lightweight, nonmagnetic, nonconducting cylindrical carrier. The magnet carrier operates between an inner and outer laminated Hyperco stator. The submodule output coils are connected in series. Thermal power is supplied to the SPDE by pumping a hot (~630 K) molten salt heat-transfer fluid through the shell of each tube-in-shell heater unit. Similarly, thermal power is rejected from the module to water circulated through the shell of each tube-in-shell cooler. The electrical output of the module is rectified to dc and is dissipated through a 25-kW_o resistance unit.

The SPDE is designed for steady-state operation at the design point and, as such, has no control system. Variations in power are accomplished through temperature changes in the external heating/cooling systems and load applied to the system.

Table 1 presents SPDE design parameters; Table 2 lists the SPDE materials; and Table 3 presents the SPDE engine geometry. A more complete description of the design is contained in Brown (1987)*. Major hardware components of the SPDE, as fabricated, are shown in Figures 2 through 9. Typical engine build sheets for the SPDE are provided in Appendix A.

The SPDE initial assembly and initial hot low-pressure run were performed in June 1985. As shown in Figure 10, the engine was installed in the test cell horizontally and was suspended from the ceiling by four straps, such that dynamic balance of the opposed piston configuration could be directly observed.

In early SPDE tests, the measured indicated power was =15% below design predictions at 75 bar mean pressure. However, at full pressure, 150 bar, the power was 50% below the predicted level. Disassembly of the engine showed that the stacked, unsintered regenerator screens were badly damaged. Metallurgical analysis of the screens indicated relative motion between the screens and the regenerator housing walls. Subsequent investigation revealed that the manufacturer had rolled the screens to 75% of the desired thickness, which resulted in a loose packing of the screens in the regenerator housing. The action of the reversing flow in the engine was determined to be the cause of the damage. The screens were replaced by a sintered 25µm wire felt metal regenerator with a tight fit in the regenerator housing. In addition to replacing the screens with the felt metal, wire stand-offs were added to prevent direct contact of the regenerator with the heater or cooler (i.e., provided a small plenum between the regenerator and heater and the regenerator and cooler). This change improved the flow distribution. Engine tests with the new regenerator showed good correlation with predictions at both low and high mean pressure. Details of early SPDE tests are given in Dochat (1987). Evaluation of the regenerator screen failure is contained in Hull (1987).

2.2 SPDE Testing in 1986

For thermodynamic characterization of the SPDE, engine parasitic losses were evaluated in a series of cold tests (temperature ratio = 1.0). Alternators motored the pistons with the displacers either held fixed at their top-dead-center position or free to move. With the displacers held fixed, the power needed to drive the pistons equaled the engine parasitic losses (predominately the pressure-induced hysteresis and leakage losses). A comparison of measured and predicted indicator power over ranges of mean pressure and piston stroke showed that these losses were close to predictions.

^{*} References are located following Appendix D.

TABLE 1 — SPDE DESIGN POINT OPERATING PARAMETERS

Design Parameter	Value
Frequency Mean Pressure Heater Wall Temperature Cooler Wall Temperature Displacer Stroke Amplitude (XD) Piston Stroke Amplitude (XP) Phase Angle XD Relative to XP Compression-Space Pressure Amplitude	105.0 Hz 150.0 bar 630.0 K 315.0 K 8.97 mm 10.16 mm 65° * 14.38 bar

^{*}During early testing of the SPDE, displacer gas spring volume was changed by adding a stuffer to achieve a phase angle of about 90° for maximum power.

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TABLE 2 — SPDE MATERIALS

Component	Material
Displacer Dome and Radiation Shields Support Cone Displacer Rod Gas Spring Piston and Cylinder Flange and Post Heater and Cooler Tubes Heat Exchanger Structure Power Piston and Cylinder Plunger Carrier Pressure Vessel Alternator	Inconel-718 Beryllium Beryllium/Steel Steel Inconel-718 Inconel-718 Beryllium/Steel Titanium Low-Alloy Steel Epoxy Hyperco, Copper

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TABLE 3 — SPDE ENGINE GEOMETRY

Displacer	
Hot side diameter (m):	1.143×10^{-1}
Hot side area (m ²):	1.0261×10^{-2}
Cold side area (m ²):	9.9725×10^{-3}
Maximum amplitude (m):	1.24×10^{-2}
Displacer mass (kg):	1.701
1	
Piston	1.4478×10^{-1}
Diameter (m):	1.4478×10^{-2} 1.6463×10^{-2}
Piston area (m²):	1.5403×10^{-2} 1.53×10^{-2}
Maximum amplitude (m):	
Piston mass (kg):	9.967
Heater	
Number of heater tubes:	1632
Length (m):	9.02×10^{-2}
Inner diameter (m):	1.27×10^{-3}
Wall Thickness (m):	5.08×10^{-4}
Regenerator	
Frontal area (m ²):	2.39×10^{-2}
Length (m):	2.463×10^{-2}
Wire diameter (m):	2.54×10^{-5}
Porosity (%):	83.8
Type of Matrix:	Felt metal
'-	1 011 1110 1011
Cooler	1540
Number of tubes:	1548
Length (m):	9.5×10^{-2}
Inner diameter (m):	1.52×10^{-3}
Wall Thickness (m):	5.08×10^{-4}
Cold Connecting Duct	_
Volume (m ³):	3.52×10^{-4}
Surface area (m ²):	1.00×10^{-1}
Hydraulic diameter (m):	1.37×10^{-2}
Effective length (m):	7.85×10^{-2}
Expansion Space	
Mean volume (m ³):	6.82×10^{-4}
Wetted area (m ²):	1.374×10^{-1}
·	1.57 7 × 10
Compression Space	اسمه سوري
Mean volume (m ³):	4.47×10^{-4}
Wetted area (m²):	1.11×10^{-1}
Appendix Gap	
Diameter (m):	1.143×10^{-1}
Minimum gap (m):	1.58×10^{-4}
Maximum gap (m):	3.21×10^{-4}
Effective gap (m):	2.25×10^{-4}
Effective length (m):	1.15×10^{-1}

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Figure 2. SPDE Heater Head During Assembly

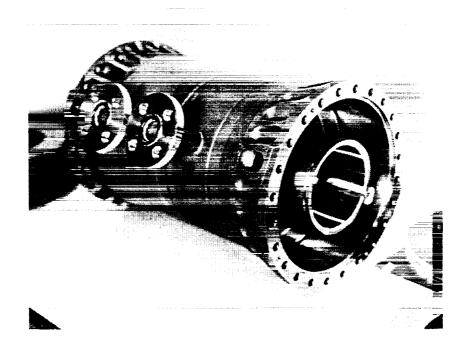


Figure 3. Completed SPDE Heater Head Assembly with Regenerator Screens

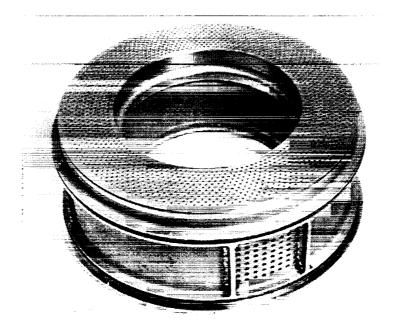


Figure 4. Completed SPDE Cooler Ready for Installation into Heater Head Assembly

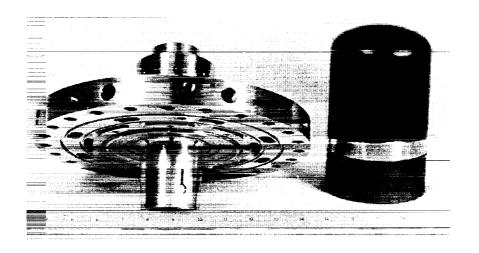


Figure 5. SPDE Post and Flange and Displacer

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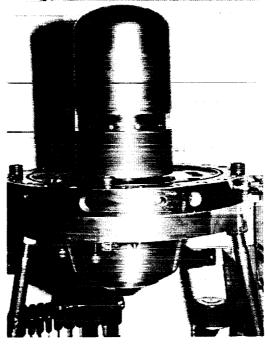


Figure 6. Completed SPDE Displacer Drive Assembly on Assembly Stand



Figure 7. SPDE Power Piston

italika akodo Hradio daji ha sindaya ako ngalaja

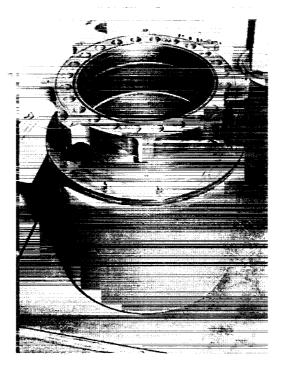


Figure 8. SPDE Power Piston Cylinder with Inner Alternator Stator Attached

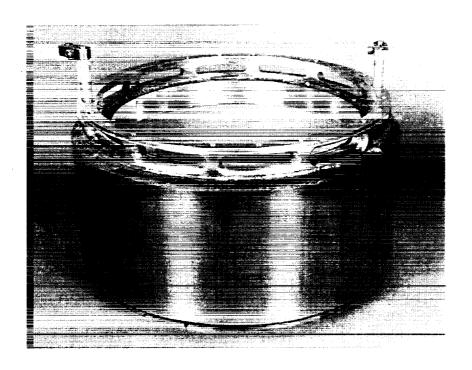


Figure 9. SPDE Alternator Stator

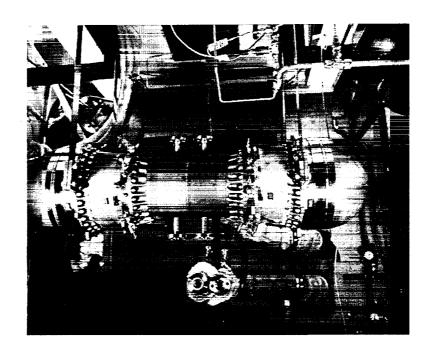


Figure 10. SPDE Installed in Test Cell

Parasitic losses associated with the moving displacer (i.e., heat exchanger pressure drop losses, flow-induced hysteresis losses, and mixing losses) were evaluated by repeating the tests with the displacer unlocked and reciprocating, and with the heat exchanger temperature ratio maintained at unity. These tests confirmed that the engine parasitic losses were close to predicted values. Details of these early parasitic loss tests are presented in Dochat (1986).

During the remainder of 1986, power module performance was characterized over a full range of operating conditions. Tests were conducted at heat exchanger temperature ratios of 1.6 to 2.0, mean pressures of 75 to 150 bar, and piston strokes of 10 to 20 mm (design stroke). Appendix B provides a summary of the data obtained after repairing the regenerator and completing the diagnostic tests. Appendix C contains selected data plots of the information contained in Appendix B. As shown in the Appendix B data and the Appendix C plots, the highest piston PV power was 25 kW which was obtained on 10/24/1986, data file SP106E, (data point 38 in Appendix B). MTI's harmonic code HFAST closely predicted engine performance. Figure 11 shows predicted and actual values for engine piston PV power, plotted as a function of power piston stroke amplitude at a temperature ratio of 2.0 and 150 bar mean pressure. Maximum measured piston PV power was 25 kW. Engine dynamics are shown in Figures 12 and 13, where the predicted and actual engine stroke ratio (defined as displacer amplitude divided by piston amplitude) and displacer-to-piston phase angle compare well.

Predictions were made with the HFAST code available at that time. The development of the MTI HFAST code is an ongoing effort supported by NASA-Lewis, and modifications are incorporated as a better understanding of loss mechanisms develops. Differences between tests and predicted data provide a starting point in our evaluation to obtain an improved analytical understanding. Complete documentation that highlights the HFAST code development history will be provided when the HFAST code is delivered to NASA. Table 4 compares HFAST to selected SPDE operating points. It is noted that the SPDE data that is presented at a temperature ratio of 2.0 is, in fact, operating closer to a temperature ratio of 2.1 due to an error in the physical fluid properties that was not corrected until after the completion of the SPDE program.

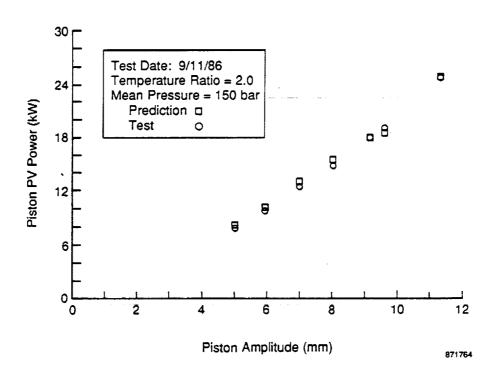


Figure 11. Hot Engine Test: Piston PV Power

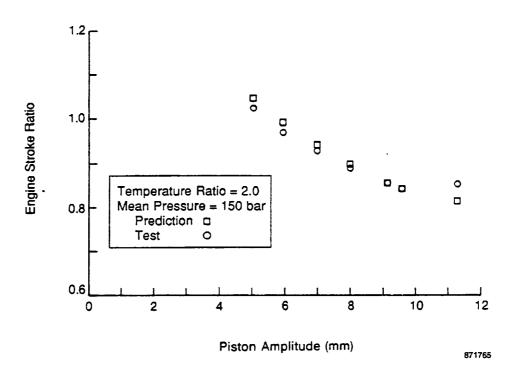


Figure 12. Hot Engine Test: Stroke Ratio

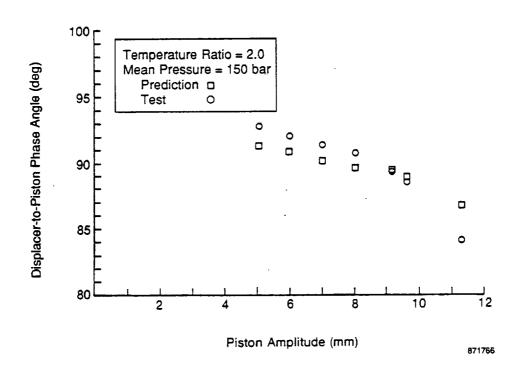


Figure 13. Hot Engine Test: Displacer-to-Piston Phase Angle

TABLE 4 — HFAST COMPARISON WITH ACTUAL PERFORMANCE AT SELECTED SPDE OPERATING POINTS

			Operating Conditions	onditions				Perf	Performance		
Scan No.	Temperature Ratio††	Mean Pressure (bar)	Frequency (Hz)	Piston Amplitude (mm)	Stroke Ratio*	Displacer Phase Angle	Pressure Ratio** (%)	Compression Space Pressure Angle	Piston PV Power (kW)	Heat In (kW)	Heat Out (kW)
27	2.018	150.3	100.7	5.01	1.02	92.83	5.85 (6.09) [†]	-9.7 (-10.34)	7.82 (8.572)	40.8 (37.2)	27.9 (28.1)
30	2.002	150.4	100.4	5.93	896:0	92.05	6.80 (7.14)	-8.77 (-9.36)	9.71 (10.76)	50.0 (46.9)	34.6 (35.4)
34	1.995	150.3	100.1	86'9	0.927	91.42	7.92 (8.35)	-8.1 (-8.66)	12.44 (13.65)	61.3 (59.4)	45.5 (44.9)
39	1.986	150.3	62'66	8.00	0.887	90.75	9.06 (9.51)	-7.45 (-7.99)	14.73 (16.40)	7.89	56.2 (54.8)
42	1.965	150.3	99.55	9.13	0.854	89.30	10.29 (10.75)	-6.95 (-7.25)	17.85 (19.16)	91.8 (87.3)	67.3 (67.0)
43	1.953	150.2	99.44	9.57	0.843	88.52	10.77 (11.22)	-6.78 (-6.95)	19.02 (20.08)	99.4 (93.4)	69.6 (72.1)

*Stroke Ratio = Displacer Amplitude
Piston Amplitude

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**Pressure Ratio = Compression Space Pressure Amplitude
Mean Pressure

† (HFAST Prediction)

†† Due to an error in coolant fluid properties, actual temperature ratios were closer to 2.1.

2.3 SPDE Accomplishments

Major SPDE accomplishments achieved by October 1986 are listed below:

- · Achieved operation at design stroke, pressure, and temperature
- Demonstrated 25-kW piston PV power versus 28.8-kW goal
- Achieved 22% piston PV efficiency versus 28% goal
- Demonstrated 17-kWp power versus 25-kWp goal
- · Demonstrated excellent dynamic balance
- Measured 0.03-mm casing motion amplitude at design point versus 0.08-mm maximum permissible
- · Achieved stable operation over entire operating range
- Obtained good data correlation with MTI's HFAST Stirling engine harmonic code.

At the completion of the SPDE testing and demonstration program, the SPDE linear alternator, as installed, was operating at 70% efficiency versus a design of 90%. Subsequent SPDE alternator bench tests indicated the electrical output power shortfall was due primarily to eddy current losses in the alternator support structure, not the alternator itself.

The SPDE was the first engine designed at a temperature ratio of 2.0. The development and validation of analytical codes were significant results of the SPDE program. Thermodynamic efficiency at 22.5% versus a 28% efficiency goal was demonstrated. Improved analytical codes indicated that changes in regenerator porosity and aspect ratio, as well as modifications to heater and cooler geometries, would significantly improve efficiency. These changes will not be made to the SPRE, but will be incorporated into the first test article in any follow-on program. Therefore, the knowledge gained during the SPDE program is directly transferable to future generations of Stirling space engines.

3.0 SPRE OVERVIEW

The SPRE module incorporates a 12.5-kW SPDE submodule with a flat closure plate welded to the pressure vessel to form the expansion space. The SPRE engine geometry is presented in Table 5. The only changes from the SPDE engine geometry are the volume and wetted area of the expansion space. HFAST was used to analyze SPRE performance; a reduction in expansion space volume, as compared to the SPDE increased engine power. Displacer gas spring seal clearances were also improved. To reduce the vibrations of the resultant single-cylinder engine, a dynamic vibration absorber was designed as an attachment to the engine. The absorber was assembled and has been successfully tested at both 75- and 150-bar mean pressure and frequency conditions. SPRE tests were conducted, however, with the engine connected to a seismic mass.

The acceptance testing conducted on SPRE-I prior to shipment to NASA-LeRC ran smoothly. The measured PV and electrical power were slightly above the highest previous data; 13 kW PV and 8 kW electrical were measured. Although there were no formal acceptance criteria required for this test, PV and electrical power of greater than one-half the SPDE power was the informal goal. This performance was achieved. This section summarizes the principal dynamic and thermodynamic performance results of the testing. A complete description can be found in Rauch, 1987.

3.1 Engine and Test Configuration

Acceptance test hardware included the SPRE-I engine, an electrical load configured for rectified ac operation, and a seismic mass coupled to the engine heater. The remaining support equipment and facilities were the same as those used for previous SPDE and SPRE tests.

The SPRE configuration is shown in Figure 14. During checkout tests, a crack developed in the displacer bearing plenum cover. This caused larger than normal midstroke bias in the displacer position and higher losses in the aft displacer gas spring. This bias restricted the operating range of stroke and pressure and was the primary problem resulting from this failure. For the final acceptance test run, the original plenum (P/N 1015C03-0121) was replaced with a stronger, redesigned plenum (P/N 1015C03-315). The new plenum reduced the aft gas spring volume by approximately 14.75 cm³.

The spool that connects the engine to the seismic mass was redesigned to correct problems identified during the previous checkout testing. The present spool is made of steel, which has a similar thermal expansion to the Inconel (Inco)-718 heater and has therefore eliminated bending stresses in the hot-end spool bolts. The spool wall thickness was reduced to address a concern that thermal stresses in the spool may cause high stresses in the welds. The hot-end spool bolts were replaced with bolts that have constant strength up to 1460 K. The mating surface of the mass was machined flat and certified grade-8 bolts replaced bolts of undocumented pedigree at the mass or cold end of the spool. All spool bolts were torqued to 60 in./lb with nuclear-grade, antiseize lubricant during assembly. The spool bolts require periodic checks to protect against loosening.

Two minor discrepancies exist in the engine hardware. First, the cooler used in the SPRE has a minor leak in one tube joint. Therefore, gas bubbles may be observed in the coolant outlet and the cooling system must be vented to prevent pressure buildup. Second, during the final assembly, new bolts were used for fixturing the flange and post to the heater head. These bolts were slightly too long and one bolt galled during removal. In the process of retapping the hole to clean it out, the tap broke, and rather than remove the tap, the remaining bolts were used for the assembly. As time permits, removal of the tap is recommended.

TABLE 5 — SPRE ENGINE GEOMETRY

	
Displacer Hot side diameter (m): Hot side area (m ²): Cold side area (m ²): Maximum amplitude (m): Displacer mass (kg):	1.143×10^{-1} 1.0261×10^{-2} 9.9725×10^{-3} 1.24×10^{-2} 1.701
Piston Diameter (m): Piston area (m²): Maximum amplitude (m): Piston mass (kg):	1.4478×10^{-1} 1.6463×10^{-2} 1.53×10^{-2} 9.967
Heater Number of heater tubes: Length (m): Inner diameter (m): Wall Thickness (m):	$ \begin{array}{c} 1632 \\ 9.02 \times 10^{-2} \\ 1.27 \times 10^{-3} \\ 5.08 \times 10^{-4} \end{array} $
Regenerator Frontal area (m ²): Length (m): Wire diameter (m): Porosity (%): Type of matrix:	2.39×10^{-2} 2.463×10^{-2} 2.54×10^{-5} 83.8 Felt metal
Cooler Number of tubes: Length (m): Inner diameter (m): Wall Thickness (m):	$ 1584 9.5 \times 10^{-2} 1.52 \times 10^{-3} 5.08 \times 10^{-4} $
Cold Connecting Duct Volume (m³): Surface area (m²): Hydraulic diameter (m): Effective length (m):	3.52×10^{-4} 1.00×10^{-1} 1.37×10^{-2} 7.85×10^{-2}
Expansion Space Mean volume (m³): Wetted area (m²):	$7.61 \times 10^{-4} \\ 1.473 \times 10^{-1}$
Compression Space Mean volume (m³): Wetted area (m²):	4.47×10^{-4} 1.11×10^{-1}
Appendix Gap Diameter (m): Minimum gap (m): Maximum gap (m): Effective gap (m): Effective length (m):	1.143×10^{-1} 1.58×10^{-4} 3.21×10^{-4} 2.25×10^{-4} 1.15×10^{-1}

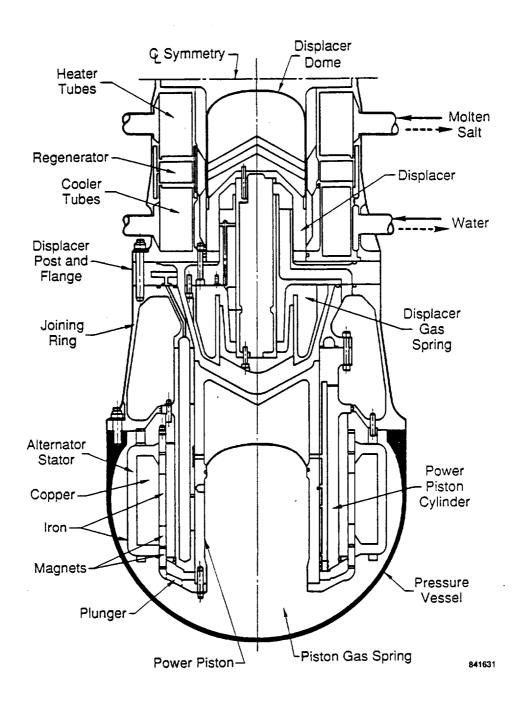


Figure 14. Space Power Research Engine (One-Half of Space Power Demonstrator Engine)

3.2 Test Procedure

The SPRE was tested following established procedures. SPRE acceptance test points are shown in Table 6. Three or more complete scans at each of the test points were obtained and then averaged to provide data. The first test point is representative of the NASA start point since NASA will use line voltage at 60 Hz to start the engine. MTI uses a power supply and therefore starts the engine at approximately 70 Hz and 75 bar.

The engine was started at a mean pressure of 75 bar and a temperature ratio of 1.6 (point 2) and then heated to a temperature ratio of 2.0 (points 3 and 4). During startup, data at temperature ratios of 1.6 and 1.8, and 10-mm piston stroke were obtained. The stroke was then varied from 10 to 18 mm at 75 bar, obtaining data points 4 through 8. The stroke was reduced to 12 mm and the pressure was increased to 150 bar, obtaining data points 10 and 11 at 100 and 125 bar, respectively. The piston stroke was again varied from 12 to 20 mm, obtaining points 12 to 16. The pressure was reduced to 75 bar and point 9 at full stroke was obtained; the 75-bar curve was repeated in descending order from point 9 to point 4. Then, as the engine was being cooled down, data at temperature ratios of 1.6 and 1.8 and 12-mm piston stroke were obtained. The pressure was then reduced to 50 bar and a temperature ratio of 1.6, (point 1), and data were obtained at a slightly varying pressure to determine the pressure for 60-Hz operation. Most of the SPRE data was obtained using externally supplied pressure for the hydrostatic gas bearings.

The tuning capacitors were switched out (in) as the pressure, and therefore, frequency was increased (decreased). The 50-bar capacitors were switched out at 62 to 63 bar as the pressure was increased to 75 bar. Table 7 shows the switches required for discrete pressure increases. When the capacitors are switched out (increasing tuned pressure), the stroke drops 1 to 2 mm; when capacitors are switched in (decreasing tuned pressure), the stroke increases 1 to 2 mm.

During cooldown, the engine was run down to a temperature ratio of 1.29 at 50 bar. Salt dilution was stopped by a high-level switch at 386 K.

3.3 Test Results

As shown in Figure 15, the peak piston PV power achieved was 13 kW at the design conditions. This value is approximately 0.6 kW above the previous record. The PV power data trends were as expected. As various operating parameters were changed, the scatter was typically less than 0.5 kW. As the design stroke was approached at 150 bar, several points were recorded at 9.5- to 10-mm piston amplitudes. Data at 75 bar and a temperature ratio of 2.0 were obtained both before and after the higher pressure operation. The amplitude was increased from 5.0 to 9.0 mm before and decreased from 10.0 to 5.0 mm after the high-pressure data were obtained. The before-and-after 75-bar data differ by approximately 0.2 kW, which is probably due to the engine having not achieved true steady-state operation before the data were taken.

Engine efficiency, based on heat rejected, is plotted against power in Figure 16. The efficiency is relatively constant at 20 to 22% for a temperature ratio of 2.0. However, there is a slight upward trend with increasing piston stroke (and consequently, PV power) at constant 75- and 150-bar levels.

TABLE 6 — SPRE ACCEPTANCE TEST POINTS

Test Point	Mean Pressure (bar)	Temperature Ratio*	Piston Stroke (mm)**
1	50	1.6	10
2	75	1.6	10
3	75	1.8	10
4	75	2.0	10
5	75	2.0	12
6	75	2.0	14
7	75	2.0	16
8	75	2.0	18
9	75	2.0	20
10	100	2.0	12
11	125	2.0	12
12	150	2.0	12
13	150	2.0	14
14	150	2.0	16
15	150	2.0	18
16	150	2.0	20

^{*}Due to an error in coolant fluid properties, actual temperature ratios were higher (2.1 instead of 2.0).

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^{**}The data tables and plots report an amplitude equal to stroke/2.

TABLE 7 — TUNING CAPACITORS REQUIRED AT VARIOUS OPERATING PRESSURES

Tuned Pressure (bar)	Switch Pressure (bar)	Total Capacitance (µf)
50	62 to 63	764
75	86 to 87	505
100	112 to 113	392
125	136 to 137	322
150	Not Switchable	292

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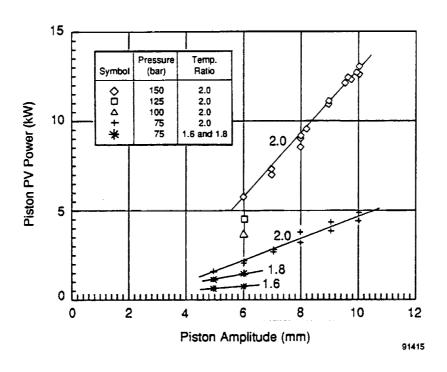


Figure 15. SPRE Test: Piston PV Power

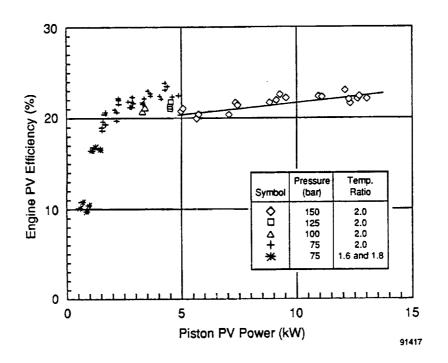


Figure 16. SPRE Test: Engine PV Efficiency

The electric output power, shown in Figure 17, generally follows the trend of PV power at a lower level. The losses in the piston gas spring and alternator account for the differences. The alternator output power plotted against piston PV power (see Figure 18) shows that the lower end efficiency is approximately 78% and 62% for 75- and 150-bar operation, respectively. Alternator efficiency at the peak power point is 70.6%. The 8.6 percentage point difference is due to power piston gas spring losses.

The acceptance test and shipment of the SPRE-I engine were successfully completed in May 1987. The engine, facility, and data acquisition all performed well, resulting in a complete set of consistent data. Also, the engine achieved the highest piston PV power and electrical output to date (13 kW and 8 kW_e, respectively).

Subsequent to delivering the SPRE-I to NASA, it was determined that the method of calculating the cooler wall temperature was in error and, hence, the temperature ratio was approximately 2.1 versus the 2.0 as reported in the test data.

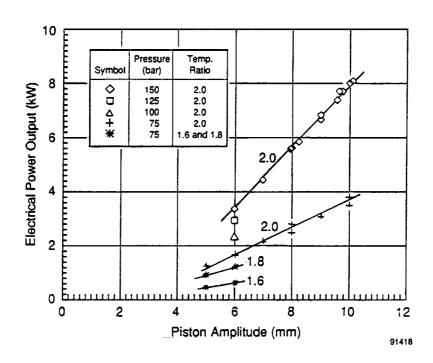


Figure 17. SPRE Test: Electrical Power Output

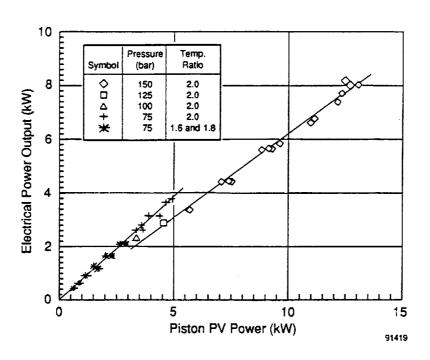


Figure 18. SPRE Test: Alternator Power vs. Piston PV Power

4.0 HYDRODYNAMIC BEARING EVALUATION

The purpose of the testing effort was to develop a hydrodynamic bearing configuration that would operate stably at engine design conditions, and to demonstrate hydrodynamic bearing operation in the running SPRE. A gas hydrodynamic bearing that operates at 150-bar mean pressure, both rotates and reciprocates, and has time-varying gas pressure waves acting on its ends (such as in an FPSE) had not been studied before.

The SPRE uses hydrostatic bearings to provide radial support for the displacer and the power piston. The term "hydrostatic" indicates that the bearing pressure profile, which generates the load-carrying capacity, is primarily the result of high-pressure-supplied bearing pressure, applied either externally or internally. Gas is introduced from a high-pressure source (i.e., higher than engine mean pressure) into the bearing clearance between the cylinder and the reciprocating piston. From this point, the gas drains to the engine mean pressure volume. The high-pressure gas in the SPRE is designed to be supplied to the bearing from the power piston gas spring. The bearing supply plenum and the piston gas spring volume are connected by means of ports when the gas spring pressure is near its maximum.

Advantages of the hydrostatic bearing include its relatively high stiffness, its high stability, and its demonstrated operation. Disadvantages are its mechanical complexity and its significant impact on engine efficiency. Mechanical complexity arises from the need for numerous drillings, orifices, and supply and drain galleries. Engine efficiency is reduced because of the high-pressure amplitude requirement in the gas springs (approximately 7 bar), which results in significant thermal hysteresis and seal leakage loss.

Hydrodynamic bearings have the potential to simplify the bearing mechanical arrangement and reduce the losses mentioned above. The bearing load capacity (i.e., bearing pressure distribution) is generated by rotational motion of the bearing journal and, therefore, does not require an external pressure source. The disadvantage of the hydrodynamic bearing is that it is susceptible to whirl instability. The primary geometric and operating variables that affect stability are bearing surface geometry, clearance, rotational speed, load, journal mass and mass moment of inertia, and bearing end conditions.

Plain cylindrical bearings operating with no load are unstable. Bearing stability increases with an increase in load, and decreases with an increase in mean pressure and rotational speed. The instability of a plain cylindrical bearing is exhibited by an increase in the journal radial displacement at half the journal rotational frequency (i.e., half-speed whirl). Half-speed whirl instability can be eliminated by incorporating herringbone grooves on the surface of the journal or the cylinder. This technique has been proven effective for journals that only rotate and do not reciprocate.

MTI investigated implementation of a hydrodynamic gas bearing on the SPRE power piston. After an in-depth literature search, as well as consultation with MTI and world-renowned bearing experts (Professor Jorgen Lund of the University of Denmark, and Professor Coda Pan of Columbia University), a detailed experimental effort was laid out. This effort involved:

- Evaluation of plain cylindrical and herringbone groove bearings in a rig that imposed SPRE operating conditions
- Selection of the preferred bearing configuration based on the rig test results
- Demonstration of the selected bearing configuration in the running SPRE at design operating conditions.

The following subsections summarize results obtained from hydrodynamic bearing tests. A complete report of the hydrodynamic test evaluation is contained in Spelter (1989).

4.1 Test Results

Tests were performed both on the bearing test rig and in the running SPRE. A hydrodynamic bearing configuration that would operate stably at engine design conditions was developed on the bearing test rig. Operation of the hydrodynamic power piston bearing was demonstrated in a running SPRE.

4.1.1 Rig Tests

Hydrodynamic tests without piston reciprocation showed stable operation up to 165 bar mean pressure. At all operating mean pressures, a half-speed subharmonic component was seen in the proximity probe measurements of the radial motion of the power piston. Although half-speed subharmonic components were present in the proximity probe signals (indicating the presence of half-speed whirl), there was no indication from the spin motor current wave form of any continuous or intermittent contact between the piston and the cylinder.

Hydrodynamic tests with piston reciprocation were performed next. Initial tests with Cylinder No. 1 (see Figure 19), which has no bearing feed holes or drain grooves present, and a plain piston showed that half-speed whirl was again present. The power piston stroke could not exceed 12 mm at 75 bar mean pressure without spin motor stall. At a mean pressure of 150 bar, the spin motor stalled when the piston stroke reached 11 mm.

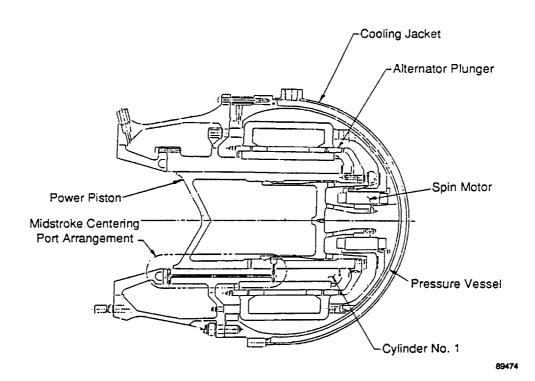


Figure 19. Cylinder No. 1 Configuration

Using Cylinder No. 1 and a piston with herringbone grooves, 50% more stroke was achieved before motor stall occurred. In addition, no half-frequency subharmonic component was seen in the proximity probe signals at lower strokes (the half-frequency component was measured just before motor stall).

Based on several diagnostic tests performed, it was concluded that instability at large strokes was due to the large pressure gradient across the bearing length caused by the compression space and power piston gas spring pressure amplitudes. Based on these results, a potential stable hydrodynamic bearing configuration was identified as a plain journal and a cylinder with circumferential grooves to isolate the bearing region from the pressure fluctuations at the ends.

Tests with a plain beryllium piston and Cylinder No. 2, which has mean pressure grooves to isolate a greater length of the bearing from pressure variations (see Figure 20), showed stable operation at design conditions out to 20-mm stroke. Although stable bearing operation was achieved with this configuration, the frequency spectrum of the proximity probe signal still showed a half-speed subharmonic component.

4.1.2 Engine Tests

Using the plain piston and Cylinder No. 2 configuration in the SPRE, stable hydrodynamic bearing operation was achieved in the running engine mode at the design operating conditions (i.e., engine heat exchanger temperature ratio = 2, mean pressure = 150 bar, operating frequency = 103 Hz, and piston stroke = 20 mm).

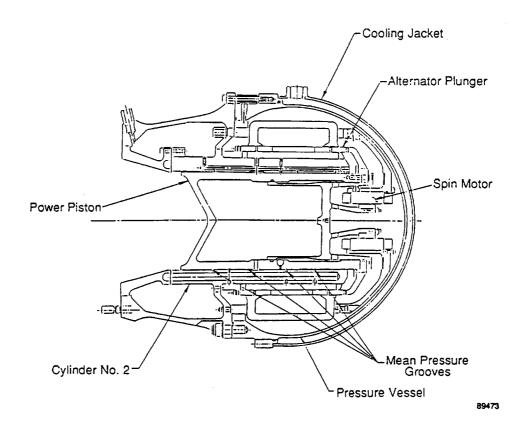


Figure 20. Cylinder No. 2 Configuration

4.2 Conclusions

A power piston hydrodynamic gas bearing operating at 150-bar mean pressure, reciprocating at 103 Hz and with 20-mm stroke (full design stroke), and rotating at 730 rpm can operate stably in the running SPRE.

An important factor in achieving stable operation at design stroke was isolating a portion of the bearing length from pressure variations. A time-varying pressure gradient across the bearing length can destabilize hydrodynamic bearing operation.

The overall hydrodynamic bearing configuration loss was approximately 700 W, compared to 1700 W for the hydrostatic bearing configuration. Table 8 contains a detailed breakdown of the losses. The potential to reduce losses by designing for hydrodynamic bearings versus hydrostatic bearings is primarily due to the reduction in the power piston gas spring hysteresis loss with the hydrodynamic bearings, which amounted to 320 W for the hydrodynamic bearings versus 1500 W for the hydrostatic bearings. This was made possible by designing the piston gas spring with a 3-bar pressure amplitude versus the 7-bar pressure amplitude required for internally pumped hydrostatic bearings.

TABLE 8 — HYDRODYNAMIC VS. HYDROSTATIC BEARING LOSSES

(Losses (in watts) at 150-bar mean pressure and 20-mm stroke)

Loss Mechanism	Hydrostatic	Hydrodynamic
Seals	176	176 ¹
Gas Spring/Porting	1500	320 ²
Rotation-Induced Losses (viscous, windage, and alternator eddy current)	0	69
Spin Motor Power	0	130 ³
Bearing Flow Power	50	0
Total	1726	695

¹Seal loss can be reduced further by designing for a longer seal length.

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²Gas spring loss is reduced by designing the piston gas spring with 3-bar pressure amplitude versus the 7-bar pressure amplitude required for internally pumped hydrostatic bearings.

³Spin motor power can be reduced to 100 W by operating at 1-ampere draw at 600 rpm.

4.3 Recommendations

Although the hydrodynamic bearing operation on the SPRE power piston was successfully demonstrated in a running engine mode, hydrodynamic bearing development needs to be continued for the following reasons:

- The SPRE hydrodynamic bearing with Cylinder No. 2, although showing stable operation, has not been optimized for minimum loss. The optimum seal length must be both long enough to minimize seal leakage loss and short enough to provide an adequate stability margin during stroking. The effort required to determine the above optimized length needs to be continued.
- An unloaded plain cylindrical hydrodynamic bearing is inherently unstable.
 Unlike the power piston bearing, which is loaded by alternator side loads, the
 displacer does not have any inherent side-loading mechanism. The displacer
 bearing, as presently configured in the SPRE, can be unstable under micro gravity space operating conditions. Therefore, displacer bearing technology
 development must be conducted.
- The spin motor currently used, a permanent-magnet synchronous motor, was
 selected on the basis of immediate availability. Based on overall system
 considerations, an induction motor will be superior because it requires
 potentially simpler controls and is less sensitive to system transients. A
 suitable induction motor has been identified and is available from Walco
 Industries.

5.0 SPRE LINEAR ALTERNATOR DYNAMOMETER EVALUATION

The SPDE alternator performed with lower-than-expected efficiency during testing. The maximum electrical power achieved was 17 kW compared to the 25-kW design goal. Subsequent SPDE alternator bench tests and a detailed finite element analysis identified problems in two areas: high magnetic permeability of the engine structure adjacent to the alternator, and excessive flux density in the inner stator at high alternator current levels. The first item results in the generation of eddy currents and corresponding eddy current losses in the adjacent structure. The second item results in local flux saturation in the inner stator at high alternator current levels and a corresponding decrease in alternator force generation, power output, and efficiency.

Finite element analysis and alternator bench tests indicated that the structural eddy current losses were the primary cause of the alternator efficiency shortfall. To verify this preliminary conclusion, additional SPDE alternator tests were performed on MTI's linear alternator dynamometer. The complete test report is contained in Rauch (1990). The specific objectives of the tests were to:

- · Evaluate alternator performance with a nonmagnetic adjacent structure
- Evaluate alternator performance in a simulated engine configuration (i.e., with a magnetic adjacent structure)
- Generate alternator performance maps to validate the alternator design and analysis methodology.

5.1 Test Results

The following alternator tests were performed on the linear alternator dynamometer:

- · Locked plunger test
- Alternator open-circuit voltage test
- Alternator performance test with nonmagnetic adjacent structure
- · Alternator performance test with magnetic adjacent structure.

The following subsections present the test results and compare the measured data to code predictions.

5.1.1 Locked Plunger Test

Locked plunger tests were performed on the SPDE alternator with a nonmagnetic adjacent structure (aluminum joining ring and cylinder). These tests were performed by setting the plunger stroke to zero amplitude and passing ac current at 60 Hz through the coil. The input current and the alternator terminal voltage were measured. A comparison of measured versus predicted voltage at various coil current levels is shown in Figure 21.

The experimental and predicted values compare well over the range of current excitation, and both indicate the fall-off of terminal voltage at high current levels due to alternator inner stator saturation.

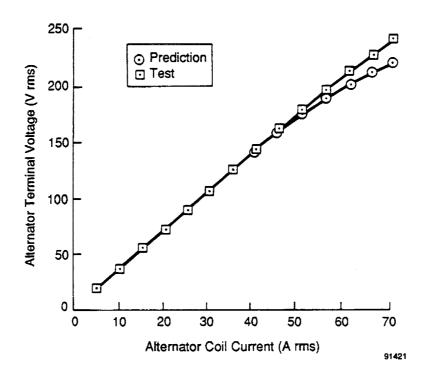


Figure 21. Locked-Plunger: Voltage vs. Current (Frequency = 60 Hz)

5.1.2 Open-Circuit Voltage Test

Open-circuit voltage tests were performed at 10.55-mm plunger amplitude, under no load (open circuit), and at various operating frequencies. Both the operating frequency and alternator terminal voltage were measured. Figure 22 shows the comparison of the predicted and measured alternator terminal voltage. As shown in the figure, the comparison is good.

5.1.3 Alternator Performance Test with a Nonmagnetic Adjacent Structure

To conduct the alternator performance test with a nonmagnetic adjacent structure, the SPDE alternator was tested on the dynamometer with the aluminum joining ring and aluminum cylinder. The alternator dynamometer tests were performed without the pressure vessel to allow for forced-air cooling. Alternator performance was measured with operating frequencies of 60 to 97 Hz, coil current of 0 to 70 A rms, and plunger amplitudes of 10.55 mm, 8.53 mm, and 6.28 mm. Most data were obtained at 10.55-mm amplitude. Due to resonance of the alternator support structure on the dynamometer, it was not possible to generate reliable test data above 90 Hz operating frequency. The choice of parameter range was based on SPDE operating conditions. The SPDE was designed with 10.16-mm plunger amplitude, 100-Hz operating frequency, and 66-A rms alternator terminal current.

Figure 23 shows the alternator efficiency versus coil current for 10.55-mm amplitude and for frequency varying from 50 to 70 Hz. The operating frequency was increased with coil current to maintain approximately zero relative phase angle between the plunger velocity and the coil current to simulate engine operating conditions with resistive load. Figure 24 shows the plot for operating frequency varying from 80 to 87 Hz.

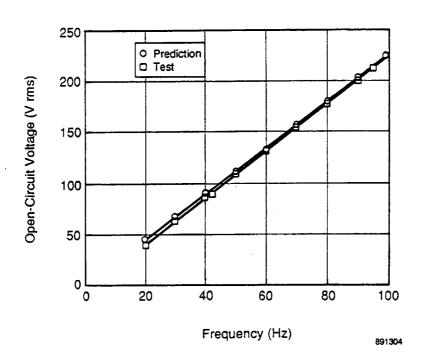


Figure 22. Alternator Open-Circuit Voltage at 21-mm Plunger Stroke

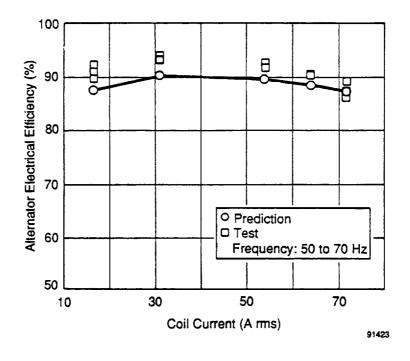


Figure 23. Predicted vs. Measured Alternator Efficiency with Nonmagnetic Adjacent Structure

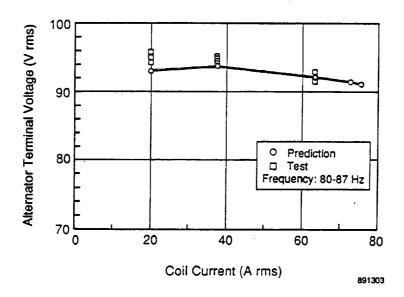


Figure 24. Predicted vs. Measured Alternator Performance with Nonmagnetic Adjacent Structure at Frequencies to 87 Hz

Figures 23 and 24 also compare the measured alternator efficiency to the code predicted efficiency. As anticipated, the measured efficiency for the SPDE alternator with a nonmagnetic adjacent structure is above 90%. As shown, the measured efficiency is close to predictions at high coil currents, and is higher than predictions at moderate currents. This behavior may be because structural eddy current losses were not calculated at all operating points. The structural eddy current losses were calculated using an axisymmetric finite element code, "FLUX," at 74 A rms coil current. At lower current levels, the eddy current losses were scaled down by the square of the value of the current.

5.1.4 Alternator Performance Test with Magnetic Adjacent Structure

To perform the alternator performance test with a magnetic adjacent structure, the SPDE alternator was tested on the dynamometer with the steel joining ring (engine joining ring is Inco-718) and cylinder. Figure 25 compares the measured performance of the SPDE alternator with magnetic and nonmagnetic adjacent structures. This figure shows the large improvement in alternator performance by replacing the magnetic adjacent structure (steel) with nonmagnetic material (aluminum). This figure also shows the alternator efficiency measured during the engine test at operating conditions of frequency = 100 Hz, plunger amplitude = 10 mm, and coil current = 66 A rms.

The analytical predictions were made using an MTI-developed alternator code, Linear Permanent Magnet Motor and Alternator (LPMMA) design and analysis code, and FLUX, a commercial finite-element software package for electromagnetic analysis. LPMMA is a time-stepping code based on a two-dimensional magnetostatic field theory, which calculates magnetic flux density distribution, magnetic force on the plunger, inductance, terminal voltage, mechanical and electrical power, magnet and lamination eddy current losses, lamination hysteresis loss, alternator coil de and ac losses, and alternator efficiency. The LPMMA does not model the leakage-flux-induced eddy current losses in the alternator adjacent structures. These losses were calculated using the FLUX code.

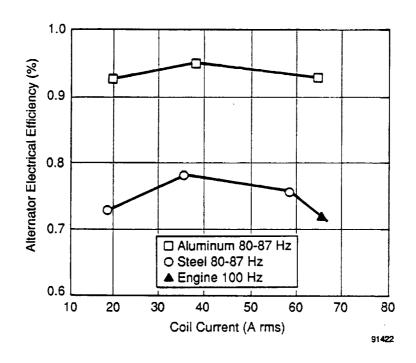


Figure 25. Alternator Performance Comparison with Magnetic and Nonmagnetic Adjacent Structure

5.2 Conclusions

The dynamometer test verified the alternator performance previously measured from the SPDE engine tests and alternator bench tests. The test results confirmed the following:

- The electrical output power shortfall in the SPDE is primarily due to eddy current losses in the magnetic structure adjacent to the alternator
- SPDE alternator efficiency of greater than 90% was demonstrated by replacing the magnetic adjacent structure with nonmagnetic material
- · Code predictions compare well with the measured dynamometer test data.

The alternator was subsequently tested in the engine with a beryllium power piston cylinder and Inco-718 joining ring and pressure vessel (nonmagnetic structure). Engine test results confirm the significant improvement in alternator efficiency and, hence, electrical power output. The maximum electrical power generated was 11.4 kW_e. Appendix D contains the high-efficiency alternator SPRE test results, including build sheets, data plots, and data summary tables.

6.0 SPRE HEAT PIPE HEATER HEAD

Integrating the FPSE into a total space power conversion system will require proper interface of the heat input to the engine and the system heat source. For a nuclear reactor system, a potential method to connect the FPSE with the primary reactor loop is via heat pipes. A heat pipe heater head was designed to assess the performance, fabrication, and geometric constraints of incorporating heat pipes into the FPSE. The heater head was designed to mate with the SPRE.

The proposed design of the SPRE heat exchangers (i.e., heater head, regenerator, and cooler) is shown in Figures 26 and 27. The design meets the following requirements:

- Incorporates heat pipes into the SPRE to assess the thermodynamic performance of a finned sodium heat pipe heater concept.
- Designed for a temperature of 1050 K in the hot, pressurized head and heater structure.
- Provides the capability to incorporate electric resistance heaters for initial testing. This will permit testing over a wide range of temperature ratios without raising the cold-side temperature.
- · Minimizes hardware changes from the existing engine.
- Provides a design life of 10,000 hr for test purposes. Eventually, the head must be capable of >60,000 hr of life.

6.1 Heater Head Design

The pressure boundary comprises a flat closure plate with a cylindrical vessel at the plate OD. The cold end of the cylindrical section is a gusseted flange of the same geometry as the current design. This design permits the same post and flange and displacer assemblies to be used without modification. The regenerator has a frontal area of 116.8 cm² (versus 83.8 cm² on the current design), which will result in higher thermodynamic efficiency. The larger frontal area was obtained in the same diametral envelope by making more effective use of the space between the displacer OD and the pressure vessel.

The cooler is conceptually similar to the existing SPRE cooler design, but has been redesigned to accommodate the added frontal area. It will also provide the seal between the expansion and compression space at the displacer OD.

The primary difference between the new design and the existing SPRE design is in the heater. It is formed by a set of tubular wells attached to the head in the annular region on the hot side of the regenerator. The heater has 45 wells symmetrically spaced in three rows, as shown in Figure 26. The outer surfaces of the wells are finned in the longitudinal direction to form flow passages for the engine working gas (helium). A solid stuffer with cylindrical holes that match the OD of the fins is used to confine the working gas to the passages between the fins.

Heat is supplied to the engine by either electric resistance heaters or heat pipes inserted into the wells. By closely controlling diametral and straightness tolerances on the ID of the wells and the OD of the electric heaters or heat pipes, and by using a heat transfer agent supplied by the heater manufacturer, good thermal contact can be ensured and the temperature of the heater element or heat pipe fluid can be kept to acceptable levels.

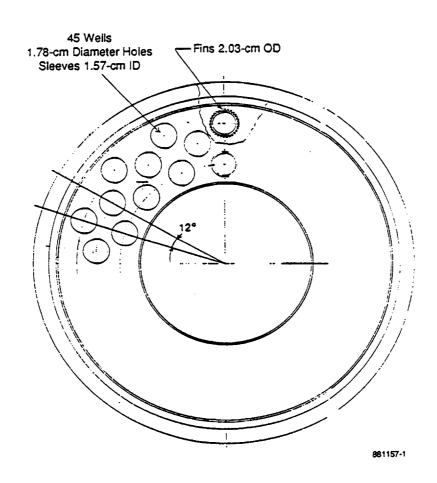


Figure 26. Top View — SPRE Head

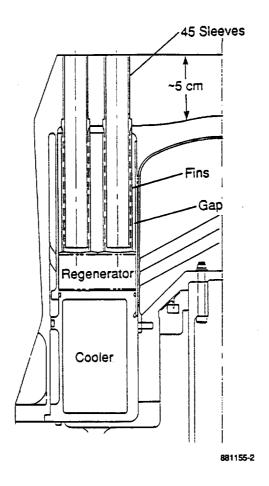


Figure 27. SPRE Heat Exchangers

Proper selection of material for the high temperature and high pressure is a critical issue. Material thicknesses and associated weights must be determined and the need (if any) to make any special provisions to control the temperature distribution of the heater head must be evaluated. Evaluations required to select the heater head material are:

- Thermal analysis to determine the temperature distribution along the vessel wall for use in determining thermal stresses and for evaluating the heat loss from the hot section
- Stress analyses to assess the structural integrity of the flat head, the vessel, and the heater wells
- Determination of assembly and fabrication procedures and associated reliability.

Listed in descending order of creep strength, the materials that have been considered for the SPRE heater head are Inco-713LC, HS-31, Inco-625, and Inco-718. Creep properties for the various materials are shown in Figure 28. The Inco-718 has higher yield and ultimate tensile strength and consequently higher fatigue strength than the other alloys.

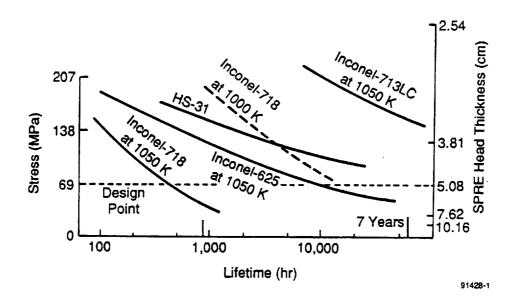


Figure 28. Material Creep Properties

6.2 Thermal Analysis

The objective of the thermal analysis was to determine the temperature distribution along the vessel wall outboard of the heater/regenerator/cooler. This distribution then was used to determine the thermal stresses in the wall. The regenerator is approximately 2.54 cm long. The operating temperature of the heater and cooler vary during testing. For design purposes, the maximum range was selected at 1050 K on the hot side and 300 K on the cold side. The gradient across the regenerator was thus 295 °C/cm. The vessel wall would be overstressed if it were located in contact with the regenerator such that it experienced the same gradient. The gap between the heater and vessel wall was incorporated into the design to help spread the thermal gradient along the wall.

A thermal model was generated to represent the high temperature gradient in the regenerator wall. One-dimensional (1-D) radial heat flow was assumed across the gap at the inner surface of the vessel wall. Conduction across the gap is formulated from gap radial conductivity and gap radial dimension. Radiation was formulated assuming constant but adjustable emissivity of the two surfaces. Again, 1-D heat flow was assumed. The 1-D heat flow was also modeled at the outer surface. Natural convection and radiation were included. The effect of incomplete insulation of this surface on the gradient and heat loss was analyzed.

Axial heat flow along the wall results from conduction in the vessel material and was determined from material thermal conductivity and wall thickness. Heat balance equations were written for elements of the wall.

Analyses in which the outer wall was not fully insulated showed that a heat loss of 3 kW was too large a penalty to pay for the small reduction in the steepness of the gradient that could be achieved. A fully insulated outer wall was selected as reference. Analyses were conducted in which the emissivity and gap conductivity were varied between the extremes of zero and maximum radial heat flow. The results of these analyses are shown in Figure 29. If heat flow across the gap could be completely suppressed, which is not physically possible, the gradient at the cold end would be 98 °C/cm and would be approximately 39 °C/cm at the hot end. At the extreme of twice nominal conduction (to allow for high convective heat transfer and for radiation based on an emissivity of 0.6, i.e., no special surface treatment to minimize it), the gradient at the cold end is approximately 157 °C/cm and the gradient at the hot end is small. The 157 °C/cm gradient will be used as reference unless stress analysis dictates that a reduction of this is mandatory to meet stress limitations.

To ensure that natural convection, which was not modeled, does not introduce serious error in the estimated gradient, circumferential fins will be machined on the shell that forms the inner surface of the annular gap.

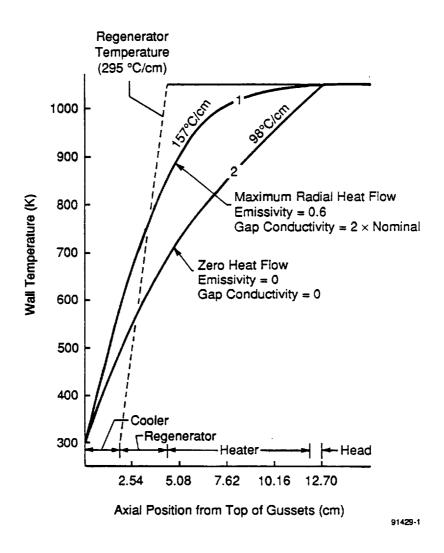


Figure 29. SPRE Wall Temperature Analysis

6.3 Stress Analysis

The dominant structural requirements on the heater head are as follows:

- Material creep of the hot section of the heater head must not result in failure or gross deformation
- Pressure cycling coupled with the effects of the thermal gradient must not result in high cycle fatigue failure
- Thermal cycling due to startup and shutdown coupled with the pressure loading must not result in low cycle fatigue failure. (This condition is usually met automatically if there is no high cycle fatigue failure.)
- Stresses due to pressure on the cold end must not produce gross yielding or rupture. This sets the thickness of the cold section of the vessel.

The stress criteria defined by practice include a factor of safety of 1.5 against failure.

Loading conditions to be considered include:

- Mean pressure 150 bar
- Thermal gradient see Figure 29
- Alternating pressure 20 bar maximum.

The vessel thickness is selected so that the membrane stresses in the wall due to pressure do not exceed Sm.* The flat head thickness is selected so that the primary bending stresses due to pressure do not exceed $1.5 \times Sm.$

6.3.1 Creep Analysis

Conservative hand calculations of the stresses in a flat plate and a cylindrical shell under a uniform pressure load were used to select the thickness of the top plate and the outer shell using HS-31 as the preliminary reference material.

For the 5-cm thick head shown in Figure 27, the strength required based on the primary bending stresses in the head being held to $1.5 \times \text{Sm}$ dictates that Sm (i.e., $2/3 \times \text{stress}$ to rupture) (see Figure 28) be no less than 68 MPa at the design temperature of 1050 K and design mean pressure of 150 bar. The right ordinate on Figure 28 shows the thickness of the flat head to give the stress level on the left ordinate at a pressure level of 150 bar. If inco-713LC were selected, the head thickness could be reduced to approximately 3.81 cm and meet the 7-yr life requirement of a space application. HS-31 could meet this requirement at a head thickness of approximately 5 cm. For Inco-625, the head would have to be even thicker (heavier) to meet the 7-yr life goal. Inco-718 has a few hundred hours of life at 1050 K and several thousand hours at 1000 K, but cannot meet the 7-yr goal.

^{*} Sm = 2/3 yield or 1/2 ultimate strength at cold end and 2/3 stress to rupture or 1% creep in design life at hot end.

6.3.2 Outer Shell Fatigue Evaluation (2-D Analysis)

Figures 30 and 31 show low-temperature and high-temperature, high-cycle fatigue diagrams generated by assuming a straight failure line between the endurance limit on the alternating stress axis and the ultimate tensile strength on the mean stress axis. These diagrams were used to design the SPRE heat pipe heater head.

The ISOPDQ 2-D, axisymmetric, finite-element code and the heater head finite-element model were used to analyze the stresses in the vessel and head. This analysis ignores the heat pipe penetrations, which were analyzed separately. This code was used for the stress analysis because it is economical and the array of gussets at the flange can be easily modeled.

Regions where stress conditions are of primary concern are:

- The point where the cylindrical section and the gusseted flange section meet (since this is where the sharp change in thermal gradient, as shown in Figure 29, occurs).
- The intersection of the vessel and flat head, where discontinuity stresses are induced by the pressure loading. The thermal stresses at this location reduce the maximum stress. However, since the gradient is small, as shown on Figure 29, this reduction is neglected.

At the cold end, thermal stress adds to the pressure stress. At the hot end, thermal stress subtracts from the pressure stress. Distribution No. 1, as shown in Figure 29, was selected as the reference for the cold end. This distribution is believed to be more severe than would be expected. The actual temperature distribution could be reduced (closer to Distribution No. 2) by controlling the emissivity with appropriate plating. It is predicted that any of the four materials could meet the fatigue limits at the cold end of the gradient. Sufficient data exists on Inco-718 to confirm that the design would be satisfactory. Additional data needs to be obtained on the other materials to verify the design. Since the thermal stresses are fairly small for Distribution No. 1, and since they subtract from the pressure stresses, they are conservatively neglected.

6.3.3 Heat Pipe Penetration Fatigue Evaluation (Three-Dimensional Analysis)

In order to evaluate the stresses around the holes in the top plate where the heat pipes (or electric heaters) penetrate the pressure boundary, a three-dimensional (3-D) finite-element analysis was performed using the ANSYS code.

Pressure loading was applied to the underside of the head and at the cylinder ID. The maximum stress in the head was in the ligament at the inner row of holes, which is what would be expected. As shown in Figure 32, the margin against fatigue is ample.

6.3.4 Heat Well Stress Analysis

The heater well is subjected to an external pressure load on the finned section below the head. The design pressure load is 150 bar mean pressure and 22.5 bar alternating pressure.

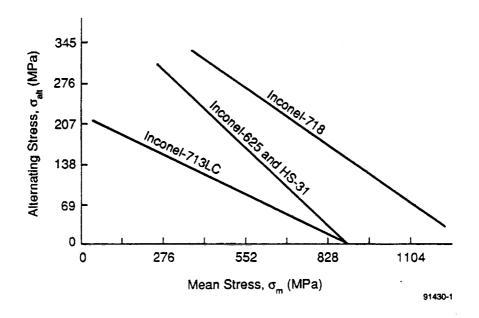


Figure 30. High-Cycle Fatigue at Low Temperature (273 to 523 K)

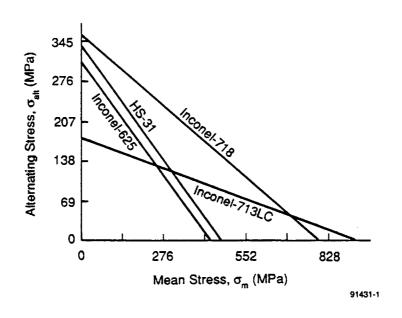


Figure 31. High-Cycle Fatigue at High Temperature (1050 K)

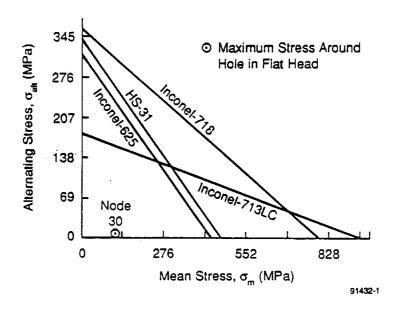


Figure 32. Predicted vs. High-Cycle Fatigue at High Temperature (1050 K)

The ID of the well is 1.57 cm, which has an interference fit with a nominal 1.58-cm diameter heater or heat pipe. The OD in the head section is selected at 1.78 cm (i.e., 0.1-cm wall) to minimize the diameter of the penetrations in the head. In the finned section, the wall thickness, and consequently the wall OD at the base of the fins, can be larger, if necessary.

At a thickness of 0.1 cm, the hoop stress in the well at a mean pressure of 150 bar is 129 MPa. At the maximum pressure of 172.5 bar, the stress is 148 MPa. These calculations neglect any load-carrying capability of the heater or heat pipe inside the well. They are compressive membrane stresses that are limited by creep and buckling, respectively. For a 0.1-cm wall, the critical buckling pressure is 565 bar. Hence, the factor of safety is 3.3, which is above the required value of 2.5.

It is conventional to assume that creep in compression has the same magnitude as in tension. This is probably a highly conservative assumption and tests are being recommended to evaluate this further, since it can have significant impact on the temperature drop across the wall of the engine heater and can affect the overall efficiency of space engine power systems.

For the SPRE, this assumption is not a critical issue since the performance tests will be conducted over a wide range of temperature levels and temperature ratios, and the temperature ratio will be referenced to the base of the fins. At 1050 K, the maximum temperature drop across the well wall adjacent to the regenerator is approximately 270 °C/cm.

Since compressive creep of the well could deform the electric heater or heat pipe, a conservative approach is to assume that compressive creep is the same as tensile creep and limit the radial deformation due to this mechanism. It is uncertain whether removal

of electric heaters following operation can be done without a machining operation. Limiting compressive creep deformation increases the chance that such removal will be possible. Replacement of heat pipes is less likely, but the same argument applies.

Figure 33 shows the calculated percent reduction in well diameter and the approximate radial deformation in mils. The high-strength casting alloys, HS-31 and Inco-713LC, show very small deformation (less then 0.00025 cm for operation of over 10,000 hr). The lower creep strength of Inco-718 and -625 results in 1% deformation (about 0.00075 cm) for a 0.1-cm thick wall at 1000 K at about 100 hr. As shown on the figure, a thicker wall (0.2 cm) can operate at 1000 K for about 1,000 hr. This is a viable option for a performance test engine, where operating hours at the peak temperature will be limited. For an endurance demonstration at 1050 K, the high-strength alloys should be selected pending further data on compressive creep.

Line	Material	Temperature (K)	Well Wall Thickness (cm)
Α	Inconel-718	1050	0.10
В	Inconel-625	1050	0.10
C	Inconel-718	1050	0.20
D	Inconel-718	1000	0.10
E	inconel-718	1000	0.20
F	HS-31	1050	0.10
G	Inconel-713LC	1050	0.10

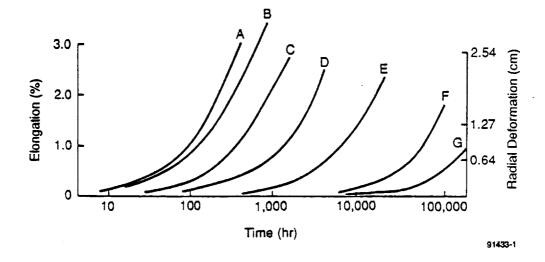


Figure 33. Creep SPRE Heater Wells Assuming Compressive Creep is the Same as Tensile Creep (Mean Pressure = 150 bar; ID = 1.57 cm)

6.4 Recommendations

Based on the analyses presented in this section, the following recommendations are made.

For the near term, an Inco-718 head should be considered since:

- · A comfortable margin against fatigue failure is predicted.
- The cost should be modest since the gusset section from an existing head could be used and the plate and cylinder sections are straightforward.
- Fabrication should be straightforward since the joining to the head of the wells and shell can be done reliably with electron-beam welds.
- The time required to procure material and fabricate the heater head will be much less than for a cast material (Inco-713LC or HS-31).
- The life at 1050 K is conservatively estimated to be 500 hr. This is less than the specified requirement. However, since the time at temperature does not need to be very long during typical performance tests, this life span should be acceptable for the first head.

A second head made of cast Inco-713LC should be considered since:

- Inco-713LC would be the material of choice for a long-life, minimum-weight design.
- This head has a 7-yr life and would permit endurance testing at a hot-side temperature of 1050 K to be performed after other performance and development tests have been completed.
- · A long-life head operating at 1100 K or slightly higher is possible.
- The joining processes are more difficult than with Inco-718 but are not expected to be extremely difficult since they are primarily for sealing.
- The Inco-713LC head and heat pipe heater arrangement, with relatively minor modifications, could be a viable candidate for a system that meets space application goals of efficiency and specific weight.

APPENDIX A SPDE INSPECTION AND BUILD SUMMARY

					-			
	Engine No: 11	Build	Stant:	<u>6</u> /	19/86	Engineer	•	
	Build No: 20	Build	Comple	te: <u>Z</u> /	17/86	Technici	an JVH/G	P II_
	Component P/N:	1015-	S/N	Design	Actual	Weight	Date	Tech
i 2	Heater (1632 tubes) Displ Cyl Seal	E 0050	1 1 1 1 1 1	(1632 4.5040	Tubes), 4.504	<u>18.035</u> Kg	到8年	SPA
3	Regenerator (1.6 mil x 200 mes	C-0119	حشيب المؤالين	54 s		<u>1.005</u> Kg	2_/25/86	$\mathcal{A}\mathcal{L}$
4567	Cooler (1584 tubes) Outer vent orifice Inner vent orifice Nuts (24)	D-0068 B-0147	, <u>-1</u> 10	0.006 0.006		9,290 Kg Kg Kg ,300 Kg	生/25/85	
8	Cooler I/O Flanges(2) Rolts (8)	-0130	182			<u>.694</u> Ка <u>.024</u> Ка	多一种是	TER
10 11	Pressure Vessel w/s Nuts (30)	D-0122	w	with outsta	ops	19745 Kg	生經盤	<u> </u>
12 13	Alt Cooling Jacket Bolts (4)	-0123	1			3.153 Kg 064 Kg	美紙器	ISB ISB
14 15 15	Disp Dome Assem Fwd G.S. Seal Disp Exp/Cmp Seal	D-0037	2 0D	3.3514 4.5000	33514	<u>1.1457</u> kg	至1248	
17 18 19	Disp Rod Bolts & Washers(4) Disp Rod Erng/Seal	D-0070	3	1.8000	17996	<u>.2775</u> K.a <u>.0282</u> K.a avg.	#/29/34 #/28/86	
20. 22. 22.	Gas Spring Piston Rolts & Washers(4) Gas Spr Piston Seal	D-0075	OD	3.2640	32651a	.2445Kg 0155 Kg	生/29/3	
23 24 5 20 20	Flange & Post w/inst Bore Brng/Seal Fwd G.S. Seal Fixture Bolts (4)	D-0113	3 00 00	1.8010 3.3500	18017 33490	12.5002Kg	生產	
26	Damper Valve Assy			<u>nstalled</u>		K.g	_/_/_	
27 28 29	Gas Spring Cylinder Bolts (E) Aft G.S. Seal Stuffer Volume	D-0106	E3-2 ID	3.2659 (<u>5.38</u>	3.2667 15**3	3 7045 K G -0509 K G	世	
30	Joining Ring w/ inst & studs	D-0112	_1_			<u>25.170</u> Kg	5/22/85	<u> 2</u> 77+
स्टाट्सटाट अभ्रत्यात्त्र	Piston Cyl w/ plugs holts (8) G.S. Erng Supply Po krng Ket Orifice Cyl Bore			Open U.0200 5.7010	<u>×</u> Clsd <u>5.7018</u>	18.350 K a — 572 K a — K a K a	五年一般	
36 37	Power Piston w/studs Pist Brng/Seal	D-0088	<u>2</u> 00	5.7000		<u>3.025</u> Kg	4/8/器	J9R MM
38 39	Plenum Cap Assy w/stiffne Bolts (9)	~~0i00	<u>.5</u> _	-		378 Kg .025 Kg	1/2/2	<u> </u>
40 41 42 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	Nuts (18) Magnet Diameter Magnet Diameter Additional Pluncer Mosa	D-0036	Z ID CD	8.360 A 8.940 A	8.330 8.970	5580 Kg -025 Kg	生性器	盤
44 45 45	Inner Alt Stator W/s Nuts & Washers Plenum Inner Stator	<i>v bjaða ju</i> D-0 950	\ <u>2</u>	8.300	5.298	-10040-Kg 6-612-Kg 010-Kg	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	JSR JSR JSR
47 48 49	Outer Alt Stator W/s : Nuts & Washers Stator Bors	D-0015	ID	9.000	8990	26.00 Kg 015 Kg	型型學	TZK TXH

	SPDE inspection & Build Summary (cont) page 2											
	Engine No: 11-ft Build Start Build No: 20 Build Compl		Engineer _ Technician	JAH/AIN								
	Component P/N: 1015- S/N	Design Actual	Weight	Date Tech								
50	Rearing Clearances: Displacer Rod Power Piston	0.0010 .0020 0.0010 .0015										
2004 MOV S	Seal Clearances: Disp Exp/Cmp Fwd Disp G.S. Disp Fwd Disp G.S. Rod Aft Disp G.S. Rod Aft Disp G.S. Rod Piston Cmp Space Piston Gas Spring	0.0040 042 0.0010 020 0.0010 020 0.0010 020 0.0010 020 0.0010 020 0.0010 022										
59 60	Alternator Plunger Clearances: Inner Gap Outer Gap	0.060 <u>v.030</u>										
61 62 63	Total Dynamic Mass: Piston Assem Dynamic Mass Displacer Assem Dynamic Mass Casing Assem Dynamic Mass		10.04 Kg 4	28/86 <u>All</u>								

64 Total Engine Mass

	110							
	Engine No: 2 Right	Poils 9	Stant:	6/	19/86	Engineer		
	Build No: 20	Build (Comple	te:/,	_/_	Technici	TAME UE	Ш
	Component P/N:	1015-	S/N	Design	Actual	Weight	Date	Tecn
i 2	Heater (1632 tubes) Displ Cyl Seal	E 0050	2	(<u>1477</u> 4.5040	4.504	<u>18.035</u> Kg	多一般	GDA MA
3	Regenerator (i.6 mil x 200 me	C-0119	WA	54 50 wick 2 Dut	rns)	<u>940</u> kg	2/25/86	
4567	Cooler (1584 tubes) Outer vent orifice Inner vent orifice Nuts (24)	0-0068	ID ID	0.006 0.006		9.327 Kg ————————————————————————————————————	5/20/85 5/24/85	
B	Cooler I/O Flanges(2 Bolts (8)	0130	384			.693 Kg .024 Kg	至/強/題	JSR JSR
10 11	Pressure Vessel W/s Nuts (30)		<u>2</u> _v	rith outs	tops	19.42 Kg 555 Kg	生性僧	<u>₩</u>
12 13	Alt Cooling Jacket Bolts (4)	-0123	2_			3.105 Kg 0x4 Kg	弄凝器	JSR JSR
14 15 15	Disp Dome Assem Fwd G.S. Seal Disp Exp/Cmp Seal	D-0037		3.3514 4.5000	3.3514 4.4997	7768 Ka	5/32/35/35 2/32/35/35	THE THE
17 18 19	Disp Rod Bolts & Washers(4) Disp Rod Erng/Seal	D-0070	<u>1</u> DD	1.8000	1,8000	.220 Kg .028 Kg	至三點	
20 21 22	Gas Spring Piston Rolts & Washers(4) Gas Spr Piston Sea	D-0075	0D 153-1	3.2640	3,26530	245 Kg	五篇篇	128
234 23 23 25	Flange & Post w/inst Bore Brng/Seal Fwd G.S. Seal Fixture Bolts (4)	D-0113	1D OD	1.8010 3.3500	3.35 <u>00</u>	15557 Kg	子/22/監 子/21/監 子/11/監	JUH
26	Damper Valve Assy			nstalled		Kg	_/_/_	
27 28 27	Gas Spring Cylinder Rolts (B) Aft G.S. Seal Stuffer Volume	D-0105	ID	3.2659 (<u>5.38</u>	<u>3.2668</u>	3.434 Kg _,052 Kg >	美多器	12E
30	Joining Ring ₩/ inst & studs	D-0112	2			25.14 Kg	<u>4 / 25/85</u>	<u>JSR</u>
31 32	Piston Cyl w/ plugs kolts (B)	D-0089	1			18.37 Kg	五光器	138 <u>P</u>
33 34 35	Rolts (B) G.S. Brng Supply P Brng Ret Orifice Cyl Bore	ert Plu	g <u>- i</u> j ID	05en 0.0200 5.7010	∑C1sd 5.7014a	K a	工工工	MM
36 37	Power Piston w/studs Pist Brng/Seal		,	5.7000	5.2001	3.025 Kg	井海器	JSR MM
38 39	Plenum Cap Assy with st Bolts (9)	iffres 100	4			<u>.375</u> Kg <u>.025</u> Kg	幸/姓/醫	JVH JSR
444	Alternator Plunger Nuts (18) Magnet Diameter Magnet Diameter Additional Plunger mass	D-0036	ID OD	8.360 8.940 ი	8-330 8-330	5 553 Kg 025 Kg 10.04 Kg	41224124	盤、
44 45 45	Inner Alt Stator w/s Nuts & Washers Inner Stator	D-0020	1	8.300	8.297	<u>010</u> k² € €22 k² 10.05 k°	五度器	11 TEB
47 48 49	Duter Alt Stator W/s Nuts & Washers Stator Bors	D-0015	<u>2</u> ID	ē.000	8.9983	<u>26.61</u> Kg 015 Kg	五10 殿	JSR JSR

SHUE	inspection	4	Build	Summary	(cont)	page 2

النبن <u>ي</u>	Engine No: 2Right Build Star Build No: 20 Build Comp Component P/N: 1015- S/	lete: 7/11/86		JSR n JW/GU Date Tecn
50 51	Bearing Clearances: Displacer Rod Power Piston	0.0010 <u>00/3</u> 0.0010 <u>00/3</u>		
2134 M 67 8 M M M M M M M	Seal Clearances: Disp Exp/Cmp Fwd Disp G.S. Disp Fwd Disp G.S. Rod Aft Disp G.S. Rod Aft Disp G.S. Rod Piston Cmp Space Piston Gas Spring	0.0040		
59 60	Alternator Plunger Clearances: Inner Gap Outer Gap	0.060n <u>.330</u> 0.060n <u>.283</u>		
61 63 63	Total Dynamic Mass: Piston Assem Dynamic Mass Displacer Assem Dynamic Mass Casing Assem Dynamic Mass		10.04 Kg 1733 Kg Kg	生/超/起 删
64	Total Engine Mass		Кд	-/-/

APPENDIX B SPDE DATA SUMMARY REPORTS

NOMENCLATURE

 P_{mean} Mean Pressure (Pa) **FRQ** Frequency (Hz) TRTOA Temperature Ratio XPA Piston Amplitude (m) Displacer Amplitude (m) XDA **XDPA** Displacer Phase Angle (deg) **PVPSTS** Piston PV Power (W) KW(ALT) Electrical Power Output (W)

							_
Data Summary Rep	ort from: FRO(XPL) 4	SP086B::45	at 4:15 XPA 6	PM FRI., 3 XDA 7	OCT., 1986 XDPA 8	Plot F	'ile: SPL1A::45 KW(ALT) 10
7 0825 0601 2 5174E+06 5 0925 0743 4 4369E+06 60825 0749 5 4300E+06 10825 0756 6 4296E+06 0825 0809 7 4022E+06	0.0000E+00 0.0000E+00 0.0000E+00 0.0000E+00 0.0000E+00	1 3270E+00 1 4561E+00 1 4642E+00 1 4743E+00 1 5015E+00	0.0000E+00 0.0000E+00 0.0000E+00 0.0000E+00	0 0000E+00 0 0000E+00 0 0000E+00 0 0000E+00 0 0000E+00	0.0000E+00 0.0000E+00 0.0000E+00 0.0000E+00 0.0000E+00	0.0000E+00 0.0000E+00 0.0000E+00 0.0000E+00 0.0000E+00	5 1441E+00 8 8373E+00 4 0889E+00 3 8251E+00 4 2208E+00
6 0825 0908 7.6177E+06	7.2917E+01	1.6367E+00	4.3722E-03	3.4635E-03	9.6298E+01	1.0233E+03	7 7214E+02
7 0825 0919 7.4992E+06	7.2324E+01	1.6295E+00	4.9967E-03	3.7150E-03	9.5450E+01	7.6838E+02	9 5588E+02
10825 0937 7.4767E+06	7.1639E+01	1.6203E+00	5.0869E-03	3.7207E-03	9.9978E+01	5.2327E+02	9 7755E+02
10825 0940 7.4982E+06	7.2256E+01	1.6111E+00	4.9663E-03	3.8470E-03	9.7225E+01	7.4982E+02	8 7107E+02
10 0825 0948 7.5007E+06	7.2210E+01	1.6074E+00	4.9512E-03	3.8345E-03	9.7191E+01	1.1205E+03	8 6157E+02
11 0825 0954 7.5181E+06	7 2312E+01	1.6101E+00	4.9678E-03	3.8474E-03	9.7350E+01	1 1539E+03	8.7714E+02
12 0825 1003 7.4935E+06	7 2159E+01	1.5974E+00	5.0277E-03	3.8600E-03	9.77885+01	1 1162E+03	8.4060E+02
13 0825 1019 7.4935E+06	7 2473E+01	1.6816E+00	5.0191E-03	4.0432E-03	9.4602E+01	1 4755E+03	1.1469E+03
14 0825 1111 7.4898E+06	7 2913E+01	1.7941E+00	5.0108E-03	4.2452E-03	9.1298E+01	2 0037E+03	1.5159E+03
15 0825 1126 7.5055E+05	7 2976E+01	1.9042E+00	5.0314E-03	4.2649E-03	9.1150E+01	2 0472E+03	1.5540E+03
16 0825 1129 7 5009E+06	7 2923E+01	1.8038E+00	5.0267E-03	4.2610E-03	9 1003E+01	2 0487E+03	1.5449E+03
17 0825 1151 7 5108E+05	7 3294E+01	1.9336E+00	5.0784E-03	4.4524E-03	8 8703E+01	2 5630E+03	1.9322E+03
18 0825 1225 7 5089E+06	7 3371E+01	2.0028E+00	5.0647E-03	4.5164E-03	8 7324E+01	2 8497E+03	2.0873E+03
19 0825 1229 7 5152E+06	7 3379E+01	2.0048E+00	5.0618E-03	4.5165E-03	8 7470E+01	2 8673E+03	2.0878E+03
20 0825 1254 7 5045E+06	7 3228E+01	1.9988E+00	6.0204E-03	5.1780E-03	8 6754E+01	3 7562E+03	2.7886E+03
21 0825 1256 7 5011E+06	7 3182E+01	1.9985E+00	6.0291E-03	5.1927E-03	8.67835+01	3.7554E+03	2 7905E+03
22 0825 1258 7 4953E+05	7 3173E+01	1.9998E+00	6.0292E-03	5.1971E-03	8.6562E+01	3.8040E+03	7909E+03
23 0825 1320 7 5080E+06	7 3123E+01	1.9981E+00	7.0449E-03	5.8897E-03	9.5972E+01	4.8735E+03	3 6211E+03
24 0825 1322 7 5069E+05	7 3131E+01	2.0001E+00	7.0574E-03	5.8928E-03	9.6006E+01	4.874E+03	62448E+033
25 0825 1326 7 4805E+06	7 3032E+01	2.0108E+00	6.1045E-03	5.1948E-03	8.6971E+01	3.5917E+03	7597E+03
26 0825 1327 7.4608E+06 27 0825 1339 7.5126E+06 28 0825 1341 7.5039E+06 29 0825 1348 7.4880E+06 30 0825 1350 7.5015E+06	7 2878E+01 7 2980E+01 7 3013E+01 7 2813E+01 7 2887E+01	2.0061E+00 2.0065E+00 2.0065E+00 1.9996E+00 2.0005E+00	7.0043E-03 8.0239E-03 8.0100E-03 8.9657E-03 8.9834E-03	5.8021E-03 6.3809E-03 6.3651E-03 6.9594E-03 6.9871E-03	8.5132E+01 8.5892E+01 B.5970E+01 B.5463E+01 B.5409E+01	4 5889E+03 5 6402E+03 5 5451E+03 6 6524E+03	3.4715E+03 4.1998E+03 4.1882E+03 4.9427E+03 4.9737E+03
31 0825 1417 9 9834E+06	8.3513E+01	2.0082E+00	6.0720E-03	5.3194E-03	9 0229E+01	5.5000E+03	3.8015E+03
32 0825 1417 9 9845E+06	8.3519E+01	2.0076E+00	6.0759E-03	5.3292E-03	9 0180E+01	5.4939E+03	3.7982E+03
33 0825 1510 1 2540E+07	9.2665E+01	1.9953E+00	5.8581E-03	5.3594E-03	9 2818E+01	6.9622E+03	4.5017E+03
34 0825 1515 1 2550E+07	9.2767E+01	2.0033E+00	5.8191E-03	5.3455E-03	9 2818E+01	7.0700E+03	4.6088E+03
35 0825 1518 1 2548E+07	9.2656E+01	2.0033E+00	6.0554E-03	5.5156E-03	9 2332E+01	7.5435E+03	4.9432E
36 0825 1525 1.3528E+07	9.6188E+01	2.0016E+00	5.9701E-03	5.6374E-03	9.0734E+01	B. 4574E+033	5.2697E+03
27 0825 1549 1.4999E+07	1.0035E+02	2.0044E+00	6.2849E-03	5.8467E-03	9.3407E+01	5514E+033	6.3888E+03
26 0825 1551 1.4999E+07	1.0035E+02	2.0029E+00	6.3213E-03	5.8744E-03	9.3400E+01	551420E+03	6.4678E+03
27 0825 1558 1.5017E+07	1.0037E+02	2.0033E+00	6.4171E-03	5.9605E-03	9.3171E+01	51420E+03	6.46137E+03
40 0825 1605 1.5039E+07	1.0028E+02	2.0010E+00	6.3083E-03	5.8752E-03	9.3344E+01	512439E+04	6.4936E+03
41 0825 1607 1.5030E+07	1.0039E+02	2.0020E+00	6.3857E-03	5.9341E-03	9.3277E+01	1.0691E+04	6 5883E+03
42 0825 1610 1.5012E+07	1.0033E+02	1.9993E+00	6.3778E-03	5.9322E-03	9.3244E+01	1.0635E+04	6 5367E+03
43 0825 1616 1.4989E+07	1.0004E+02	1.9807E+00	7.1941E-03	6.4852E-03	9.3244E+01	1.2627E+04	7 8688E+03
44 0825 1619 1.5017E+07	1.0015E+02	1.9773E+00	7.1683E-03	6.4652E-03	9.2547E+01	1.2532E+04	7 8602E+03
45 0825 1625 1.5022E+07	9.9879E+01	1.9539E+00	8.2208E-03	7.1546E-03	9.1097E+01	1.5063E+04	9 7113E+03
46 0825 1628 1.5023E+07	9 9892E+01	1.9525E+00	9.2395E-03	7.1801E-03	9 1206E+01	1.5104E+04	9 7532E+03
47 0825 1632 1.5005E+07	9 9799E+01	1.9548E+00	8.2224E-03	7.1681E-03	9 1180E+01	1.4957E+04	9 7209E+03
48 0825 1634 1.4978E+07	9 9723E+01	1.9561E+00	8.2476E-03	7.1850E-03	9 1154E+01	1.5117E+04	9 8612E+03
49 0825 1637 1.4992E+07	9 9782E+01	1.9607E+00	8.2421E-03	7.1858E-03	9 1154E+01	1.5004E+04	9 8744E+03
50 0825 1639 1.5023E+07	9 9847E+01	1.9607E+00	9.2137E-03	7.1967E-03	9 1322E+01	1.5131E+04	9 9296E+03
\$1 0825 1644 1 5015E+07 \$2 0825 1647 1 5008E+07 \$3 0825 1651 1 4999E+07 \$4 0825 1658 1 5012E+07 \$4 0825 1658 1 5022E+07 \$6 0825 1700 1 4973E+07 \$6 0825 1700 1 5042E+07 \$6 0825 1700 1 5042E+07 \$6 0825 1700 1 5042E+07 \$6 0825 1700 1 5042E+07 \$6 0825 1700 1 5042E+07	7 7853E+01 7 7871E+01 9 7871E+01 9 7826E+01 9 7826E+01 9 7816E+01 9 7818E+01 9 7675E+01 0 0000E+00	1 7728E+00 1 7822E+00 1 7821E+00 1 7821E+00 1 7821E+00 2 7821E+00 2 01555E+00 2 1555E+00 1 0555E+00	8 21454E-03 8 2454E-03 9 2455E-03 9 2455E-03 8 2337E-03 9 2455E-03 0 0000E+00	7 2077E-03 7 2050FE-03 7 24458E-03 7 1837E-03 7 1837E-03 7 2295E-03 7 8296E-03 0 0000E+00	9 1475E+01 9 1373E+01 9 1478E+01 9 1474E+01 9 1774E+01 9 0571E+01 0 0000E+00	1.5155E+04 1.5630E+04 1.5719E+04 1.5745E+04 1.5745E+04 1.5765E+04 1.70655E+04 1.8555E+04 0.0000E+00	1 0085E+04 1 0225E+04 1 0247E+04 1 0450E+04 1 0202E+04 1 0202E+04 1 1310E+04 1 5169E+01 3 2975E+00

No.	Date		Summarv Rep PMEAN	ort from: FRO(XPL)	SP096A::45 TRTDA	at 3:37 F	PM FRI., 3	OCT., 1986 XDPA	Plot f	ile: SPLig::45 KW(ALT)
	004 0	4 N 2 A	5.0563E+06 6.1012E+06 7.0306E+06 7.6187E+06 7.3773E+06	6.3452E+03 9.2023E+02 4.4725E+03 1.0792E+03 1.5201E+03	1 6232E+00 1 6266E+00 1 6304E+00 1 6333E+00 1 6234E+00	2.6297E-06 1.7793E-06 3.4740E-06 4.0578E-07 5.6954E-04	1.2445E-06 2.9453E-07 1.9821E-06 2.0114E-06 5.4405E-04	1 1046E+02 8 8696E+01 -2 9465E+01 1 6833E+02 1 2499E+02	2 6775E-01 8 2986E-02 -6 5455E-02 -9 2882E-02 -8 8775E+02	6.5950E+00 4.7484E+00 5.8036E+00 7.2545E+00 8.9837E+02
6 7 8 9	0910 0910 0911 0911 0911	1154 1203 1110 1112 1120	7.5120E+06 7.4981E+06 7.3978E+06 7.4931E+06 7.4899E+06	1.4891E+03 7.2216E+01 7.1639E+01 7.2057E+01 7.1960E+01	1.6305E+00 1.6396E+00 1.6305E+00 1.6233E+00 1.6052E+00	7.8840E-04 5.0131E-03 4.7782E-03 4.9872E-03 4.9941E-03	5.8920E-04 4.2293E-03 3.9444E-03 4.1048E-03 4.0899E-03	8.8980E+01 9.2515E+01 9.3195E+01 9.3411E+01 9.4123E+01	1.7933E+02 1.4360E+03 -5.5527E+00 1.3227E+03 1.1843E+03	1.1231E+03 1.145BE+03 9.7896E+02 1.0522E+03 9.8819E+02
	0044	4424	7 5089E+06 7 5006E+06 7 5157E+06 7 5127E+06 7 5210E+06	7.2074E+01 7.2075E+01 7.2544E+01 7.2559E+01 7.2821E+01	1.6055E+00 1.6098E+00 1.7039E+00 1.7159E+00 1.8145E+00	4.9822E-03 5.0127E-03 5.0120E-03 4.9261E-03 5.0389E-03	4 0823E-03 4 1273E-03 4 3520E-03 4 2947E-03 4 5190E-03	9.4405E+01 9.4031E+01 9.1210E+01 9.1006E+01 8.8682E+01	1.2165E+03 1.2881E+03 1.7405E+03 1.7693E+03 2.1947E+03	9.8754E+02 1.0224E+03 1.3931E+03 1.3678E+03 1.7531E+03
17	0911 0911 0911	1338 1358 1401	7.5136E+06 7.5057E+06 7.5057E+06 7.5100E+06 7.4962E+06	7 2727E+01 7 2897E+01 7 3055E+01 7 3085E+01 7 3066E+01	1.8009E+00 1.8915E+00 2.0000E+00 2.0006E+00 2.0015E+00	5.0172E-03 4.9546E-03 5.0334E-03 5.0646E-03	4.4886E-03 4.5429E-03 4.6937E-03 4.7205E-03 4.7256E-03	8 8831E+01 8 7376E+01 8 5452E+01 8 5406E+01 8 5460E+01	2 1494E+03 24625E+03 2499E+03 0011E+03 982BE+03	1.6943E+03 1.8929E+03 2.2382E+03 2.2680E+03 2.2649E+03
223345	0911	1501	1.0001E+07 1.2509E+07 1.2503E+07 1.4972E+07 1.5048E+07	8.3498E+01 9.2473E+01 9.2562E+01 1.0034E+02 1.0063E+02	2.0035E+00 2.0028E+00 2.0020E+00 1.9990E+00 2.0119E+00	5.0635E-03 5.0747E-03 5.1121E-03 5.2246E-03 5.2066E-03	4 9342E-03 5 1362E-03 5 0488E-03 5 2575E-03 5 2714E-03	8.8661E+01 9.0361E+01. 9.1035E+01 9.3037E+01 9.2845E+01	4 6560E+03 6 5473E+03 6 0785E+03 7 8923E+03 8 2053E+03	3.1907E+03 4.0541E+03 3.9364E+03 4.8033E+03 4.8525E+03
22223	0044	4540	1.5042E+07 1.5031E+07 1.5024E+07 1.5035E+07 1.5044E+07	1:0064E+02 1:0065E+02 1:0069E+02 1:0037E+02 1:0037E+02	2.0169E+00 2.0172E+00 2.020BE+00 2.0011E+00 2.0011E+00	5.1719E-03 5.0097E-03 4.9645E-03 5.9962E-03 5.928BE-03	5.2578E-03 5.1142E-03 5.0279E-03 5.7996E-03 5.7399E-03	9.2630E+01 9.2829E+01 9.2662E+01 9.2000E+01 9.2051E+01	8.2580E+03 7.8150E+03 7.4885E+03 9.7056E+03	4.7841E+03 4.5331E+03 4.4082E+03 6.0389E+03 5.8838E+03
323 333 35 35	0911 0911 0911 0911 0911	1555 1604 1605 1607 1611	1.5041E+07 1.5037E+07 1.5041E+07 1.5034E+07 1.5035E+07	1.0041E+02 1.0012E+02 1.0010E+02 1.0008E+02 9.9837E+01	1.9998E+00 1.9951E+00 1.9927E+00 1.9944E+00 1.9794E+00	5.9368E-03 7.0182E-03 6.9396E-03 6.9758E-03 8.0073E-03	5.7546E-03 6.5089E-03 6.4454E-03 6.4663E-03 7.1626E-03	9 2022E+01 9 1314E+01 9 1451E+01 9 1419E+01 9 0440E+01	9.6704E+03 1.2608E+04 1.2303E+04 1.23436E+04 -6.5902E+01	5 9831E+03 7 8470E+03 7 7854E+03 7 8295E+03 9 7272E+03
36 37 38 39 40	0911 0911 0911 0911	1613 1615 1617 1621 1624	1.5034E+07 1.5034E+07 1.5032E+07 1.5032E+07 1.5044E+07	9 9832E+01 9 9867E+01 9 9836E+01 9 9793E+01 9 9573E+01	1.9804E+00 1.9811E+00 1.9811E+00 1.9853E+00 1.965E+00	7 9880E-03 7 9737E-03 7 9878E-03 7 9924E-03 8 8980E-03	7.1629E-03 7.1322E-03 7.1677E-03 7.0880E-03 7.6610E-03	9.0396E+01 9.0635E+01 9.0532E+01 9.0532E+01 9.0749E+01 8.9485E+01	1.5046E+04 1.4893E+04 1.4957E+04 1.4957E+04 1.7356E+04	9.7502E+03 9.7574E+03 9.8523E+03 9.9156E+03 1.1550E+04
41 42 43 44 45	0911 0911 0911 0915 0915	1627 1628 1632 0905	1.5030E+07 1.5029E+07 1.5023E+07 7.5933E+06 7.6095E+06	9.9534E+01 9.9545E+01 9.9436E+01 0.0000E+00 7.2344E+01	1.9642E+00 1.9643E+00 1.9524E+00 1.6002E+00 1.5841E+00	9.0938E-03 9.1275E-03 9.5714E-03 0.0000E+00 5.3104E-03	7.7618E-03 7.7912E-03 8.0658E-03 0.0000E+00 4.2027E-03	8 9212E+01 8 9296E+01 8 8517E+01 0 0000E+00 9 4716E+01	1.7567E+04 1.7853E+04 1.7851E+04 0.0000E+00 1.2124E+03	1:1787E+04 1:1913E+04 1:2824E+04 5:6717E+00 9:4454E+02
47 48	0915	0934	7.4852E+06 7.5007E+06 7.4972E+06 7.6031E+06 7.5170E+06	7.2014E+01 7.2187E+01 7.1983E+01 0.0000E+00 7.2277E+01	1.6070E+00 1.6113E+00 1.6027E+00 1.6312E+00 1.6086E+00	4.9983E-03 5.0601E-03 6.0155E-03 0.0000E+00 5.0030E-03	4.0617E-03 4.206E-03 4.8205E-03 0.000E+00 4.0522E-03	9 4307E+01 9 3567E+01 9 2527E+01 0 0007E+00 9 5049E+01	1.1909E+03 1.3031E+03 1.6105E+03 0.6000E+00 1.1847E+03	1 2997E+03 3 4294E+00
52	0918 0918 0918 0918 0918 0918 0918	1023 1039 1049 1054 1055	7 5021E+06 7 4970E+06 7 5036E+06 7 5131E+06	7 2567E+01 7 2809E+01 7 3047E+01 7 3195E+01 7 3195E+01 7 3088E+01 7 39910E+01 7 33392E+01 8 3368E+01	1.7017E+00 1.8072E+00 1.9087E+00 1.9087E+00 2.0101E+00 2.0101E+00 2.0165E+00 2.0176E+00 2.0170E+00 2.0167E+00	5 0062E-033 4 9168E-033 4 9164E-033 4 9644E-033 6 0069E-033 6 0069E-033 6 9644E-033 6 9645E-033 7 9545E-03	4 2476E-03 4 3920E-03 4 4408E-03 5 5226-03 5 3492E-03 6 01393E-03 5 5168E-03	9 2119E+01 8 9299E+01 8 7337E+01 8 7537E+01 8 4527E+01 8 4190E+01 8 3796E+01 8 7725E+01 8 7927E+01	5832E+03 0731E+03 2237918E+03 37918E+03 37918E+03 3 9058E+03 4 992E+03 4 9923E+03 5 8448E+03 5 8448E+03	1 6236E+03 1 8562E+03 2 1005E+03 2 9481E+03 2 9728E+03 3 8077E+03 3 8308E+03
5.	0918 0918 0918 0918 0918	120	3 1.2498E+07 4 1.2522E+07 2 1.5010E+07 5 1.5000E+07 8 1.4972E+07	9.2340E+01 9.2422E+01 1.0025E+02 1.0027E+02 1.0010E+02	2 0095E+00 2 0118E+00 2 0066E+00 2 0077E+00	6.0556E-03 6.0645E-03 6.0645E-03 5.7845E-03 5.7812E-03	5.7256E-03 5.7413E-03 5.9025E-03 5.8364E-03 5.7422E-03	9.0141E+01 9.0086E+01 9.1168E+01 9.1055E+01 9.1206E+01	7.9560E+03 5.8030E+03 1.0489E+04 5.2018E+03 5.0528E+03	5.4302E+03 6.4233E+03 6.1033E+03
6657	0918 7 0918 3 0918 9 0918 0 091	1 123	4 1 2625E+07 3 9 3926E+06 7 7 5125E+06 8 7 5299E+06 2 7 3643E+06	9.2645E+01 8.2665E+01 7.2687E+01 7.1933E+01 7.0157E+01	2.0114E+00 1.9870E+00 1.9521E+00 1.9898E+00 1.5934E+00	5 8324E-03 6 0141E-03 4 7068E-03 6 1578E-03 6 3298E-03	5.5820E-03 5.4008E-03 4.2704E-03 4.8969E-03 4.3506E-03	9.0413E+01 8.9961E+01 8.9096E+01 9.4837E+01 1.0277E+02	3.9856E+03 5.4361E+03 2.4463E+03 1.3983E+03 1.3426E+03	5.0379E+03 3.8156E+03 1.8002E+03 2.5003E+03 1.0042E+03

No Da	Data te Time 2	Summarv Rep PMEAN	ort from: FRG(XPL)	SP096B::45 TRTQA	at 3:45 XPA	PM FRI., 3	OCT., 1986	Plot f PVPSTS	Ile: Shirt: 4 KW(ALT)
1 099 3 099 4 099 5 099	29 1003	7.6397E+06 7.5376E+06 7.4940E+06 7.5161E+06 1.0008E+07	0 0000E+00 7 2072E+01 7 2155E+01 7 2189E+01 8 2454E+01	1.6545E+00 1.6171E+00 1.6376E+00 1.6366E+00 1.6214E+00	0.0000E+00 6.7157E-03 4.9521E-03 5.0006E-03 5.0059E-03	0 0000E+00 5 3337E-03 4 1783E-03 4 2280E-03 4 3996E-03	0 0000E+00 9 2838E+01 9 3602E+01 9 3492E+01 9 5855E+01	0.0000E+00 2.0663E+03 1.3841E+03 1.3911E+03 1.9746E+03	4 .0889E+00 1 .5536E+03 1 .0965E+03 1 .1153E+03 1 .5104E+03
6 09: 7 09: 9 09: 10 09:	29 1031 29 1033 29 1052	9.9949E+06 1.2476E+07 1.2494E+07 1.3659E+07 1.5034E+07	8.2413E+01 9.1268E+01 9.1310E+01 9.5187E+01 9.9475E+01	1.6211E+00 1.6009E+00 1.6017E+00 1.5975E+00 1.6083E+00	4.9879E-03 5.0039E-03 5.0095E-03 5.0437E-03 4.9940E-03	4.3828E-03 4.5720E-03 4.5906E-03 4.7316E-03 4.8140E-03	9 5798E+01 9 6399E+01 9 6503E+01 9 5911E+01 9 5905E+01	-4 8758E+01 2 4666E+03 2 4806E+03 2 7778E+03 3 1418E+03	1.4894E+03 1.7586E+03 1.8028E+03 1.9677E+03 2.2265E+03
11 09 12 09 13 09 14 09 15 09	29 1107 29 1111 29 1113 29 1116 29 1117	1.5052E+07 1.4983E+07 1.5030E+07 1.5031E+07 1.5046E+07	9.9252E+01 9.9160E+01 9.9248E+01 9.9222E+01 9.9524E+01	1.5957E+00 1.6022E+00 1.6002E+00 1.6033E+00 1.6179E+00	5.9548E-03 5.6191E-03 5.9624E-03 6.0290E-03 5.0294E-03	5.5142E-03 5.1192E-03 5.3434E-03 5.3906E-03 4.7275E-03	9.42895+01 9.57915+01 9.53445+01 9.53285+01 9.52545+01	3.9008E+03 3.3281E+03 3.5798E+03 3.7245E+03 3.1030E+03	2.6760E+03 22.1828E+03 22.3667E+03 22.4570E+03 22.0442E+03
17 093 18 093 19 093	29 1141 29 1204 29 1230	1.4972E+07 1.5078E+07 1.5107E+07 1.5052E+07 1.5047E+07	9.9280E+01 9.9765E+01 9.9935E+01 1.0021E+02 1.0030E+02	1.6732E+00 1.7076E+00 1.8113E+00 1.9243E+00 1.9968E+00	6.3183E-03 6.0096E-03 6.0074E-03 6.0132E-03 5.9847E-03	5.7317E-03 5.5758E-03 5.7273E-03 5.8645E-03 5.8997E-03	9.3475E+01 9.3575E+01 9.2544E+01 9.1505E+01 9.1465E+01	5.3215E+03 5.5985E+03 -9.0102E+01 9.2518E+03 1.0213E+04	3.6559E+03 3.6855E+03 5.0286E+03 5.0285E+03 5.3257E+03
21 092 22 093 23 093 24 093 25 093	29 1254 29 1308	1.5047E+07 1.5057E+07 1.5061E+07 1.5053E+07 1.5054E+07	1.0032E+02 1.0031E+02 1.0039E+02 1.0035E+02 9.9786E+01	2.0008E+00 2.0018E+00 2.0449E+00 2.0606E+00 2.0252E+00	6.0235E-03 6.0392E-03 6.0837E-03 6.0515E-03 7.9295E-03	5.9303E-03 5.9511E-03 5.9944E-03 5.9902E-03 7.2942E-03	9.1323E+01 9.1160E+01 9.1421E+01 9.1352E+01 9.217E+01	1.0520E+04 1.0393E+04 1.1161E+04 1.1147E+04 1.5368E+04	6.3920E+03 6.4354E+03 6.6062E+03 5.6757E+03 1.0257E+04
25 073 27 073 28 073 29 073 30 073	29 1325 29 1330 29 1336	1.5046E+07 1.5103E+07 1.5042E+07 1.5059E+07 1.5081E+07	1.0032E+02 9.9730E+01 1.0036E+02 9.9559E+01 9.9415E+01	2.0687E+00 2.0000E+00 2.0692E+00 2.0198E+00 1.9816E+00	5.9018E-03 8.9905E-03 5.9907E-03 8.9258E-03 1.0017E-02	5.8731E-03 7.9967E-03 5.8569E-03 7.8998E-03 8.4677E-03	9 1512E+01 8 7667E+01 9 1613E+01 8 8601E+01 8 6859E+01	1.0576E+04 1.9152E+04 1.0339E+04 1.8627E+04 2.1059E+04	6.4653E+03 1.2204E+04 6.3312E+03 1.2105E+04 1.3871E+04
31 093 32 093 33 093 34 093 35 093	29 1450 30 0915 30 0920 30 0934 30 0936	7.1507E+06 7.3165E+06 7.5464E+06 1.0013E+07 1.0008E+07	0.0000E+00 7.0932E+01 7.2232E+01 8.2136E+01 8.2138E+01	2.1488E+00 1.6126E+00 1.6271E+00 1.6103E+00 1.6101E+00	0.0000E+00 6.3720E-03 5.0245E-03 5.9530E-03 5.9281E-03	0.0000E+00 4.9112E-03 4.1258E-03 4.9813E-03 4.9701E-03	0.0000E+00 9.4410E+01 9.4614E+01 9.6090E+01 9.6183E+01	0.0000E+00 -6.6025E+01 1.3703E+03 2.3377E+03 2.3178E+03	4.7484E+00 1.3536E+03 1.0379E+03 1.8049E+03 1.7888E+03
36 093 37 093 38 093 39 093 40 093	30 1002 30 1004 30 1040 30 1042 30 1052	1.2476E+07 1.2484E+07 1.5029E+07 1.5037E+07 1.5034E+07	9.0896E+01 9.0946E+01 9.9054E+01 9.9073E+01 9.9105E+01	1.6059E+00 1.6048E+00 1.6095E+00 1.6095E+00 1.6171E+00	5.9678E-03 5.9443E-03 5.9622E-03 5.9620E-03 5.9336E-03	5.2639E-03 5.2500E-03 5.5347E-03 5.3498E-03	9.5975E+01 9.6054E+01 9.5048E+01 9.4900E+01 9.5957E+01	3.1866E+03 -3.0207E+01 4.1186E+03 4.1965E+03 3.8164E+03	2.3059E+03 2.3028E+03 2.8814E+03 2.8787E+03 2.5549E+03
41 09: 42 09: 43 09: 44 09: 45 09:	30 1055 30 1115 30 1117 30 1138 30 1139	1.5028E+07 1.5057E+07 1.5047E+07 1.5051E+07 1.5051E+07	9 9096E+01 9 9510E+01 9 9512E+01 9 9802E+01 9 9796E+01	1.6175E+00 1.7100E+00 1.7177E+00 1.8076E+00 1.8184E+00	5.9427E-03 5.9982E-03 6.0565E-03 5.9715E-03 6.0111E-03	5.3499E-03 5.5679E-03 5.6157E-03 5.6695E-03 5.7132E-03	9.6069E+01 9.4545E+01 9.4545E+01 9.3401E+01 9.3093E+01	3.8579E+03 5.4350E+03 5.6763E+03 7.0345E+03 7.2650E+03	2 S895E+03 3 8395E+03 3 8651E+03 4 7874E+03 4 9100E+03
46 097 47 097 48 097 49 097 50 097	30 1157 30 1159 30 1223 30 1225 30 1242	1.5053E+07 1.5049E+07 1.5045E+07 1.5032E+07 1.5036E+07	1.0003E+02 1.0003E+02 1.0032E+02 1.0023E+02	1 8998E+00 1 9022E+00 1 9961E+00 2 0083E+00 2 0058E+00	5.9999E-03 5.9997E-03 5.9908E-03 6.0363E-03 6.9885E-03	5.7940E-03 5.7912E-03 5.8940E-03 5.9321E-03 6.6567E-03	9.2337E+01 9.248E+01 9.1949E+01 9.1951E+01 9.0941E+01	8.5242E+03 8.6264E+03 9.8212E+03 1.0258E+04 1.2582E+04	5 5465E+03 5 5928E+03 6 1518E+03 6 2398E+03 8 1533E+03
52 09 53 09 54 09	30 1249 30 1251 30 1254 30 1256 30 1300	1.5045E+07 1.5037E+07 1.5037E+07 1.50374E+07 1.5044E+07 1.50445E+07 1.50035E+07 1.50035E+07	1.0004E+02 1.0016E+02 1.0016E+02 1.0016E+02 1.0017E+02 1.0017E+02 1.0017E+02 1.0017E+02 1.0016E+02 1.0016E+02	200647400 001159 EE+00 001159 EE+00 01159 EE+00 01159 EE+00 01159 EE+00 0116445E+00 0117435E+00 0117435E+00	7 00935 6 96578E-033 6 94578E-03 6 94578E-03 6 94578E-03 6 97458E-03 6 97454E-03 6 9194E-03 7 9009	6 6790E-033 6 6391E-033 6 5394E-033 6 53944E-033 6 4943E-033 6 4414E-033 6 4417E-033 7 1719E-03	9 .08592+01 9 .10192+01 9 .14544E+01 9 .2053E+01 9 .1573E+01 9 .1595E+01 9 .1741E+01 9 .1746E+01 9 .0470E+01	1 2551E+04 1 2423E+04 1 1910E+04 1 0734E+04 1 1879E+04 1 1879E+04 1 1421E+04 1 1322E+04 1 1347E+04 1 4385E+04	8 2373E+03 7 9052E+03 7 90523E+03 7 7553E+03 7 7553E+03 7 7553E+03 7 7553E+03 7 7573E+03 7 7589E+03 7 8882E+03
61 09: 62 09: 63 09: 64 09: 65 09:	30 1341 30 1349 30 1356 30 1415 30 1425	1.5036E+07 1.5052E+07 1.5077E+07 1.5040E+07 1.5056E+07	9 9709E+01 9 9508E+01 9 9528E+01 9 9603E+01 9 9586E+01	2.0078E+00 1.9883E+00 1.9823E+00 2.0115E+00 2.0120E+00	B 9294E-03 1 0024E-02 1 0000E-02 9 0064E-03 9 1564E-03	7.7646E-03 8.4281E-03 8.4207E-03 7.7805E-03 7.9398E-03	8.8898E+01 8.7226E+01 8.7794E+01 8.9347E+01 8.9155E+01	1.5606E+04 1.9279E+04 1.9319E+04 1.6912E+04 1.7209E+04	1.1249E+04 1.3154E+04 1.3157E+04 1.1596E+04 1.2078E+04
66 09 67 09 68 09 69 09	30 1430 30 1439 30 1444 30 1520	1 S051E+07 1 4902E+07 1 4943E+07 7 5210E+06	9.9470E+01 0.0000E+00 0.0000E+00 0.0000E+00	2.0051E+00 2.1949E+00 2.2559E+00 2.2860E+00	9.3615E-03 0.0000E+00 0.0000E+00 0.0000E+00	8.0678E-03 0.0000E+00 0.0000E+00 0.0000E+00	8.8880E+01 0.0000E+00 0.0000E+00 0.0000E+00	1.7930E+04 0.0000E+00 0.0000E+00 0.0000E+00	1.2688E+04 7.1226E+00 7.6502E+00 3.6932E+00

Data Summary Rep	ort from: SP106A:	:45 at 8:00	AM FRI. 3	3 ОСТ., 1786	Plot f	rile: SPL1D::45
No Date Time PMEAN	FRQ(XPL) TRTQA	XPA		ХДРА	PVPŞTS	KW(ALT)
1 1002 0953 7 7056E+06 2 1002 0958 7 3867E+06 3 1002 1002 7 5493E+06 4 1002 1004 7 5485E+06 5 1002 1009 7 5472E+06	0.0000E+00 1.6786E 7.1503E+01 1.6547E 7.2251E+01 1.6547E 7.2223E+01 1.6547E 7.2293E+01 1.6527E	+00 0.0000E+00 +00 5.8202E+00 +00 4.9987E-03 +00 5.0155E-03 +00 4.9666E-03	0.0000E+00 3.4.1083E-03 4.1188E-03 4.11752E-03	0 0000E+00 9 3525E+01 9 4704E+01 9 4612E+01 9 4666E+01	0.0000E+00 1.4512E+03 1.2255E+03 1.2011E+03 1.1525E+03	4.3527E+00 1.1895E+03 9.5416E+02 9.678BE+02 9.3504E+02
6 1002 1033 1 0014E+07 7 1002 1100 1 2483E+07 8 1002 1115 1 5058E+07 9 1002 1120 1 5055E+07 10 1002 1140 1 5037E+07	8 2338E+01 1 6431E 9 1191E+01 1 6284E 9 9371E+01 1 6058E 9 9355E+01 1 6056E 9 9071E+01 1 6032E	+00 5.0241E-03 +00 5.0191E-03 +00 4.9353E-03	7	9.6455E+01 9.7030E+01 9.6930E+01 9.7464E+01 9.6052E+01	1.7545E+03 2.2683E+03 2.6597E+03 2.4770E+03 3.2149E+03	1:3414E+03 1:6524E+03 1:8271E+03 1:6668E+03 2:1786E+03
11 1002 1156 1 5033E+07	9 8837E+01 1 6013E	+00 7.0048E-03	8 6.0583E-03	9.4283E+01	4.3300E+03	2.9871E+03
12 1002 1200 1 5043E+07	9 8667E+01 1 5995E	+00 7.9522E-03	6.7383E-03	9.2146E+01	5.2437E+03	3.4736E+03
13 1002 1213 1 5056E+02	9 8514E+01 1 5958E	+00 9.0345E-03	7.5603E-03	8.9410E+01	6.6841E+03	4.0917E+03
14 1002 1222 1 5036E+07	9 9312E+01 1 6946E	+00 5.9796E-03	6.2232E-03	9.2761E+01	6.5025E+03	4.5958E+03
15 1002 1243 1 5064E+07	9 9714E+01 1 7964E	+00 6.9886E-03	6.3865E-03	9.1651E+01	8.7629E+03	6.1877E+03
16 1002 1246 1 5035E+07 17 1002 1251 1 5056E+07 18 1002 1252 1 5051E+07 19 1002 1255 1 5086E+07 20 1002 1301 1 5059E+07	9.9651E+01 1.8093E 9.9552E+01 1.8030E 9.9527E+01 1.8062E 9.9459E+01 1.7922E 9.9364E+01 1.7950E	+00 7.9369E-03 +00 7.9693E-03 +00 8.9191E-03	3 7.6740E-03	9.1336E+01 8.968E+01 8.9687E+01 8.8462E+01 8.8462E+01	8.9408E+03 1.1048E+04 1.1049E+04 1.2670E+04 1.2601E+04	6.3788E+03 7.7175E+03 7.8483E+03 8.7245E+03 8.7762E+03
21 1002 1303 1 5036E+07	9.9316E+01 1.7959E	+00 8 97225-03	7.6980E-03	8.8520E+01	1.2911E+04	8.9110E+03
22 1002 1305 1 5007E+07	9.9691E+01 1.8413E	+00 6 9818E-03	6.3663E-03	9.1750E+01	9.5789E+03	5.5570E+03
23 1002 1337 1 5057E+07	1.0015E+02 1.9888E	+00 7 0108E-03	6.6411E-03	9.0345E+01	1.2595E+04	8.108BE+03
24 1002 1341 1 5066E+07	1.0023E+02 2.0013E	+00 6 9929E-03	6.6082E-03	9.0423E+01	1.2365E+04	8.1204E+03
25 1002 1359 1 5045E+07	1.0018E+02 2.0147E	+00 6 9342E-03	6.5880E-03	9.0569E+01	1.2455E+04	8.0537E+03
26 1002 1406 1 5050E+07	1.0017E+02 2.0161E	+00 5.9700E-03	6 6611E-03	9.0325E+01	1.2826E+04	8 2598E+03
27 1002 1410 1 5063E+07	1.0026E+02 2.0169E	+00 6.9322E-03	6 5256E-03	9.098E+01	1.2101E+04	7 8065E+03
28 1002 1441 1 5036E+07	1.0027E+02 2.1073E	+00 7.0007E-03	6 6110E-03	9.0763E+01	1.3425E+04	8 3812E+03
29 1002 1445 1 5057E+07	1.0021E+02 2.1096E	+00 7.3969E-03	6 6619E-03	9.0478E+01	1.4651E+04	9 3483E+03
30 1002 1448 1 5063E+07	1.0007E+02 2.1008E	+00 7.9434E-03	7 2307E-03	9.0081E+01	1.6298E+04	1 0332E+04
31 1002 1450 1 5036E+07 32 1002 1519 1 5035E+07 33 1002 1539 1 5000E+07 34 1002 1545 1 5003E+07 35 1002 1547 1 5043E+07	9 9967E+01 2 0996E 9 9839E+01 2 0035E 9 9522E+01 2 0246E 9 9503E+01 2 0226E 9 9318E+01 1 9817E	►00 7 9 1136-07	し ウ イヴՈマF=Ոマ	9.0118E+01 8.7659E+01 8.3851E+01 8.8872E+01 8.6429E+01	1.6222E+04 1.4145E+04 1.6050E+04 1.6387E+04 1.9799E+04	1 0284E+04 9 4254E+03 1 0679E+04 1 0918E+04 1 3828E+04
36 1002 1559 1 5028E+07 37 1002 1605 1 5038E+07 38 1002 1607 1 4977E+07 39 1002 1617 1 4831E+07 40 1002 1618 1 4814E+07	9 9323E+01 2 0275E 9 9905E+01 2 0635E 9 9801E+01 2 0509E 9 8581E+01 1 9438E 9 8456E+01 1 9157E	+00 9.6408E-03 +00 7.2666E-03 +00 7.1247E-03 +00 9.8248E-03 +00 1.0090E-02	6.8497E-03 6.7261E-03 8.5109E-03	8 7032E+01 9 0669E+01 9 0991E+01 8 6567E+01 8 5863E+01	1.9751E+04 1.3034E+04 1.2259E+04 1.7845E+04 1.7775E+04	1 3268E+04 8 5201E+03 8 2060E+03 1 2271E+04 1 2443E+04
41 1002 1621 1 4766E+07	9.8244E+01 1.8789E	+00 1.0082E-08	8.5588E-03	8 .6052E+01	1.6468E+04	1.1707E+04
42 1002 1625 1 4712E+07	9.8198E+01 1.8488E	+00 1.0008E-08	8.3313E-03	8 .7759E+01	1.4766E+04	1.0573E+04
43 1002 1628 1 4671E+07	9.7827E+01 1.8178E	+00 1.0029E-08	8.4166E-03	8 .6963E+01	1.4026E+04	1.0252E+04
44 1002 1631 1 4612E+07	9.7679E+01 1.8017E	+00 1.0055E-08	8.3831E-03	8 .6963E+01	1.3245E+04	9.6470E+03
45 1002 1632 1 4586E+07	9.8053E+01 1.8361E	+00 8.0094E-03	6.8788E-03	9 .1687E+01	9.6348E+03	7.0230E+03
46 1002 1637 1 4501E+07	9 7526E+01 1.8138E	+00 8.5434E-03	7 3539E-03	8 9877E+01	1.0558E+04	7 5647E+03
47 1002 1641 1 4418E+07	9 7150E+01 1 7934E	+00 8.8915E-03	7 5490E-03	8 9432E+01	1.0644E+04	7 5691E+03
48 1002 1649 1 4236E+07	9 6397E+01 1 7409E	+00 9.0702E-03	7 6140E-03	8 9297E+01	9.6739E+03	6 6402E+03
49 1002 1705 1 3875E+07	9 5107E+01 1 6935E	+00 8.4422E-03	7 0237E-03	9 1415E+01	7.1065E+03	4 8372E+03
50 1002 1713 1 2419E+07	9 0427E+01 1 6928E	+00 7.9636E-03	6 5941E-03	9 2515E+01	5.3799E+03	3 8076E+03
\$1 1002 1722 1 0008E+07 \$2 1002 1734 7 5378E+06 \$3 1002 1745 7 4734E+06 \$4 1002 1750 7 5024E+05 \$5 1002 1818 7 5727E+06 \$5 1002 1825 7 5712E+06 \$6 1002 1837 7 7350E+06 \$9 1002 1837 7 7350E+06	8 1927E+01 1 7014E 7 0969E+01 1 6753E 7 0697E+01 1 6328E 7 0698E+01 1 6098E 7 08982+01 1 5098E 7 0873E+01 1 5978E 7 0489E+01 1 4933E 7 0748E+01 1 4739E 0 0000E+00 1 5110E	+00 1.1666E-02 +00 1.1992E-02 +00 1.0943E-02	8.0963E-03 8.1552E-03 7.4607E-03	9 25201E+01 9 25201E+01 9 2415E+01 9 2415E+01 9 2431E+01 9 2431E+01 9 2431E+01	3.9112E+033 4.2533E+033 3.3433E+033 3.0023E+03 1.9349E+03 6.8868E+02 6.8866E+00 0.0000	2.9674E+03 1891E+03 2.4680E+03 2.1021E+03 1.3660E+03 1.31627E+02 2.1027E+01 2.9018E+00

Data Summarv Rep Ng Date Tige PMEgN	ort from: SP106B: FRG(XPL) TRTGA	:45 at 8:37 XPA	7 AM FRI., 17 XDA	OCT., 1986 XDPA	Plot f	ile: SPL1E::45 KW(ALT)
1 1010 1407 7.2262E+06 2 1010 1412 7.4823E+06 3 1010 1415 7.4459E+06 4 1010 1418 7.4136E+06 5 1010 1436 7.4909E+06	7 0381E+01 1 6479E 7 1577E+01 1 6385E 7 1412E+01 1 6383E 7 1255E+01 1 6432E 7 1742E+01 1 7221E	+00 5.1250E-03 +00 5.0702E-03 +00 5.0461E-03 +00 5.0821E-03 +00 6.0438E-03	3 4 0534E-03 3 4 0300E-03 3 4 0224E-03 3 4 0515E-03 4 8737E-03	9.8058E+01 9.8157E+01 9.8248E+01 9.8045E+01 9.5135E+01	-1.1692E-01 1.0165E+03 9.9796E+02 1.0345E+03 1.8058E+03	8:1554E+02 8:1343E+02 8:0182E+02 8:2002E+02 1:5266E+03
6 1014 1008 7.5340E+06 7 1014 1018 7.5140E+06 8 1014 1038 7.5159E+06 9 1014 1046 7.5340E+06 10 1014 1106 7.4338E+06	7.1732E+01 1.6271E 7.159E+01 1.6131E 7.1461E+01 1.5954E 7.1662E+01 1.6113E 6.8429E+01 1.6424E	+00 4.9340E-0 +00 4.9754E-0 +00 4.9730E-0	3 3.8731E-03 3 3.8711E-03 3 3.9091E-03	9.8811E+01 9.9120E+01 9.9607E+01 9.9158E+01 1.1315E+02	9.2110E+02 8.6878E+02 7.8774E+02 8.8454E+02 -4.6622E+02	7.5420E+02 6.8271E+02 6.2811E+02 6.9907E+02 -7.5262E+02
11 1014 1110 7.5683E+06 12 1014 1112 7.539E+06 13 1014 1112 7.539E+06 14 1014 1137 7.5046E+06 15 1014 1141 7.4912E+06	7 1791E+01 1 7111E 7 2029E+01 1 7026E 7 1785E+01 1 7026E 7 2101E+01 1 8049E 7 2040E+01 1 9041E	+00 4.8380E-0 +00 5.0144E-0 +00 5.0631E-0	3 9232E-03 3 9984E-03 4 1087E-03 4 1087E-03 4 2291E-03	9.9719E+01 9.6553E+01 9.6553E+01 9.3758E+01 9.4073E+01	1 2077E+03 1 2257E+03 1 2508E+03 1 6402E+03 1 6098E+03	1.0282E+03 1.0018E+03 1.0299E+03 1.3942E+03 1.3392E+03
16 1014 1155 7 5020E+05 17 1014 1215 7 4947E+05 18 1014 1245 7 5067E+06 19 1014 1253 7 5116E+06 20 1014 1255 7 5035E+06	7.2326E+01 1.90218 7.2492E+01 2.00308 7.2466E+01 2.00578 7.2497E+01 2.00678 7.2394E+01 2.00208	+00 4 9739E-0. +00 4 9282E-0.	3 4 3344E-03 3 4 4547E-03 3 4 4155E-03 5 2545E-03 5 2345E-03	9.1994E+01 8.9941E+01 9.0042E+01 8.9000E+01 8.9079E+01	1.9252E+03 2.3166E+03 2.2614E+03 3.2437E+03 3.1598E+03	1 6133E+03 1 8657E+03 1 8403E+03 2 6313E+03 2 6123E+03
21 1014 1257 7 5173E+06 22 1014 1313 7 5127E+06 23 1014 1315 7 5265E+06 24 1014 1319 7 5128E+06 25 1014 1323 7 5371E+06	7.2520E+01 2.00399 7.2554E+01 2.00329 7.2375E+01 2.0025 7.2279E+01 1.9990 7.2397E+01 2.0063	+00 6.9388E-0 +00 6.9589E-0 +00 6.9863E-0	2444E-03 2457E-03 25507E-03 5507E-03 5507E-03 5507E-03	8.8913E+01 8.7962E+01 8.7961E+01 8.8370E+01 8.8325E+01	3 2450E+03 4 1286E+03 4 1756E+03 3 9955E+03 4 0102E+03	2.6284E+03 3.3822E+03 3.4082E+03 3.2569E+03 3.3103E+03
26 1014 1342 7.5091E+06 27 1014 1346 7.5037E+06 28 1014 1347 7.5085E+06 29 1014 1402 1.0058E+07 30 1014 1431 1.2530E+07	7.2231E+01 2.0076 7.2275E+01 2.0068 7.2250E+01 2.0073 8.3120E+01 2.0080 9.2048E+01 1.9998	+00 7 9568E-0 +00 7 9580E-0 +00 5 9898E-0	3 6.5516E-03 3 6.5453E-03 3 5.4786E-03	8.7377E+01 8.7297E+01 8.7357E+01 9.1389E+01 9.2481E+01	5.0487E+03 5.0249E+03 5.0262E+03 4.9897E+03 6.7994E+03	4.1323E+03 4.1369E+03 4.1311E+03 3.7441E+03 4.7881E+03
31 1014 1457 1.5000E+07 32 1014 1500 1.4962E+07 33 1014 1517 1.5018E+07 34 1014 1519 1.5027E+07 35 1014 1558 1.5045E+07	9.9878E+01 1.9931 9.9833E+01 1.9935 9.9775E+01 2.0027 9.9790E+01 2.0087 9.9902E+01 2.0077	E+00 5.9738E-0 E+00 6.9452E-0 E+00 6.9392E-0	3 5 8369E-03 3 6 5747E-03	9.2843E+01 9.31166+01 9.1766E+01 9.1659E+01 9.1809E+01	9.3314E+03 8.9030E+03 1.1737E+04 1.1926E+04 1.1593E+04	6.1692E+03 5.7355E+03 7.6801E+03 7.6593E+03 7.4181E+03
36 1014 1606 1.5013E+07 37 1014 1608 1.5011E+07 38 1014 1609 1.5010E+07 39 1014 1618 1.5000E+07 40 1014 1624 1.5003E+07	9.9769E+01 2.0066 9.9774E+01 2.0070 9.9762E+01 2.0071 9.9740E+01 2.0083 9.9643E+01 1.9852	E+00 6.9181E-0 E+00 6.9261E-0 E+00 6.9437E-0 E+00 6.9528E-0 E+00 6.9247E-0	3 6.5491E-03 6.5425E-03 3 6.5751E-03 6.57631E-03 6.5161E-03	9.1761E+01 9.1847E+01 9.1578E+01 9.1763E+01 9.1788E+01	1.1734E+04 1.1757E+04 1.1876E+04 1.1876E+04 1.1309E+04	7 5451E+03 7 5475E+03 7 5554E+03 7 55940E+03 7 3340E+03
41 1014 1629 1 4999E+07 42 1014 1636 1 5015E+07 43 1014 1642 1 5017E+07 44 1014 1650 1 4979E+07 45 1014 1659 1 4994E+07	9.9556E+01 1.9566 9.9522E+01 1.9236 9.9472E+01 1.8904 9.9259E+01 1.8585 9.9150E+01 1.8221	E+00 6.9912E-0 E+00 6.9291E-0 E+00 6.9563E-0	3 6.5313E-03 3 6.4751E-03 3 6.3931E-03 3 6.39360E-03 3 6.2827E-03	9 1919E+01 9 2574E+01 9 2994E+01 9 3459E+01 9 3851E+01	1,0835E+04 1.0367E+04 9.4185E+03 8.7910E+03 7.9729E+03	7.0729E+03 6.8839E+03 6.4152E+03 6.0421E+03 5.5609E+03
46 1014 1706 1.4999E+07 47 1014 1710 1.5024E+07 48 1014 1713 1.4988E+07 49 1014 1723 1.4995E+07 50 1014 1749 1.5004E+07	9.8747E+01 1.7687 9.8937E+01 1.7674 9.8496E+01 1.7292 9.8752E+01 1.7128 9.8144E+01 1.6214	7.7646E-0 7.3693E-0 5+00 8.6689E-0 5+00 6.8803E-0 5+00 7.9089E-0	3 6.7055E-03 3 6.4424E-03 3 7.2859E-03 4.0290E-03 5.5131E-03	9.3574E+01 9.416E+01 9.2130E+01 9.5611E+01 9.5304E+01	8.3827E+03 7.5712E+03 8.7674E+03 5.7662E+03 4.6837E+03	5.7965E+03 5.3765E+03 6.1071E+03 4.0427E+03 2.9949E+03
\$\frac{1}{2} \frac{1}{2} \frac	9 8183E+01 1 5904 9 8337E+01 1 5905 9 8389E+01 1 5790 9 8424E+01 1 5701 9 8525E+01 1 5662 9 8532E+01 1 5610	E+00 6.4138E-0 6.4475E-0 6.4205E-0 6.4205E-0 6.4205E-0 6.4205E-0 6.4205E-0 6.4205E-0 6.4205E-0 6.4205E-0 6.4205E-0 6.4205E-0 7.4205E-0 7.4205E-0 7.4205E-0	5 5998E-033 5 5998E-033 6 4 8398E-033 7 4 8484E-033 7 4 1349E-033 7 7513E-03	9 8054E+01 9 9071E+01 1 0037E+02 9 9901E+01 1 0037E+02 1 0187E+02	2.2280E+03 2.1049E+03 1.7036E+03	1 4314E+03 1 2271E+03

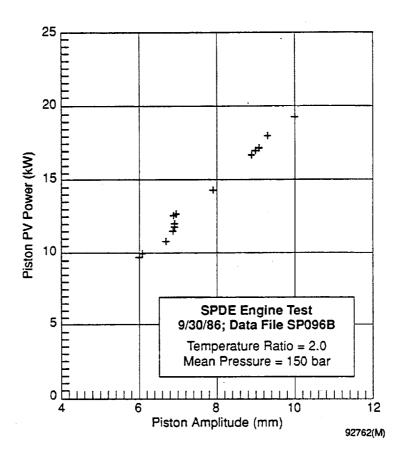
Data Summary Report from: No Date Tige PMEAN FRG(XPL)	SP106C::45	ат 9:33 ^{ХРА} 6	AM MON , 20	OCT., 1996 XDPA	Plot o	File: SPL1E::46 KW(ALT)
1 1017 1048 7.7262E+06 0.0000E+00 2 1017 1052 7.3933E+06 7.1302E+01 3 1017 1059 7.4918E+06 7.1521E+01 4 1017 1104 7.4644E+06 7.1424E+01 5 1017 1115 7.5279E+06 7.1661E+01	1.6871E+00	0.0000E+00	0 0000E+00	0 0000E+00	0 0000E+00	4 3527E+00
	1.6659E+00	4.9925E-03	4 0457E-03	9 676:E+01	1 0963E+03	8 9230E+02
	1.6510E+00	5.6307E-03	4 4301E-03	9 7698E+01	1 1989E+03	1 0329E+03
	1.6531E+00	5.2267E-03	4 0294E-03	9 8385E+01	1 1059E+03	8 4667E+02
	1.6373E+00	5.7814E-03	4 3806E-03	9 8180E+01	1 2385E+03	9 4058E+02
6 1017 1118 7 4850E+06 7 1547E+01 7 1017 1123 7 4996E+06 7 1611E+01 9 1017 1126 7 5262E+06 7 1557E+01 9 1017 1127 7 5144E+06 7 1564E+01 10 1017 1148 7 5015E+06 7 1829E+01	1.6396E+00	5.5466E-03	4 2196E-03	9.8434E+01	1.1371E+03	9.0299E+02
	1.6415E+00	5.6221E-03	4 22904E-03	9.8178E+01	1.1753E+03	9.3187E+02
	1.6382E+00	6.5792E-03	4 8743E-03	9.7264E+01	1.5031E+03	1.1779E+03
	1.6374E+00	6.5377E-03	4 8747E-03	9.7403E+01	1.5051E+03	1.1742E+03
	1.7014E+00	5.3344E-03	4 8020E-03	9.6902E+01	1.3405E+03	1.1029E+03
11 1017 1159 7 S005E+06 7 1801E+01 12 1017 1202 7 5235E+06 7 1807E+01 13 1017 1313 7 5057E+06 7 1887E+01 14 1017 1320 7 4675E+06 7 0353E+01 15 1017 1340 7 4957E+06 7 1939E+01	1.7040E+00	5.4055E-03	4 25405-03	9.5648E+01	1.3908E+03	1:1060E+03
	1.6879E+00	7.447E-03	5 255-03	9.4795E+01	2.2546	1:7846E+03
	1.7358E+00	5.3588E-03	4 255-455-03	9.6111E+01	1.4811E+03	1:1945E+03
	1.7358E+00	4.9032E-03	3 255-35	1.0695E+02	1.7211E-03	-2:3214E+02
	1.7959E+00	5.3277E-03	4 203	9.2502E+01	1.5466E+03	1:2479E+03
16 1017 1418 7 4837E+06 7 2538E+01	2.0176E+00	5.3267E-03	4.6120E-03	8 9990E+01	2.6092E+03	2.0380E+03
17 1017 1421 7 4903E+06 7 2481E+01	2.0164E+00	5.3976E-03	4.6773E-03	8 993E+01	2.6752E+03	2.0718E+03
18 1017 1423 7 4866E+06 7 2487E+01	2.0173E+00	5.3127E-03	4.6040E-03	8 9724E+01	2.5971E+03	2.0119E+03
19 1017 1425 7 5148E+05 7 2627E+01	2.0161E+00	5.3664E-03	4.6396E-03	8 9654E+01	2.5970E+03	2.0766E+03
20 1017 1432 7 4944E+05 7 2472E+01	2.0153E+00	6.3557E-03	5.3620E-03	8 8855E+01	3.5788E+03	2.7411E+03
21 1017 1435 7 5099E+05 7 2515E+01	2.0158E+00	6.3689E-03	5.3703E-03	8.8811E+01	3.5821E+03	2.7797E+03
22 1017 1439 7 5072E+06 7 2535E+01	2.0152E+00	6.4384E-03	5.4275E-03	8.8584E+01	3.6405E+03	2.8242E+03
23 1017 1447 7 5050E+06 7 2404E+01	2.0007E+00	7.4705E-03	6.1099E-03	8.7950E+01	4.6099E+03	3.5555E+03
24 1017 1449 7 4977E+05 7 2346E+01	2.0001E+00	7.3343E-03	6.0057E-03	8.8159E+01	4.4558E+03	3.55423E+03
25 1017 1450 7 5153E+06 7 2374E+01	2.0011E+00	7.5142E-03	6.1409E-03	8.7799E+01	4.6701E+03	3.5854E+03
26 1017 1500 7 5159E+06 7 2241E+01 27 1017 1502 7 5050E+06 7 2221E+01 28 1017 1504 7 5159E+06 7 2285E+01 27 1017 1507 7 5113E+06 7 2317E+01 30 1017 1509 7 5045E+06 7 2242E+01	2.0008E+00	8.7443E-03	6.9486E-03	8.6438E+01	5.9940E+03	4 S893E+03
	2.0035E+00	8.6558E-03	6.8917E-03	8.5444E+01	5.9131E+03	4 S690E+03
	2.0034E+00	8.6540E-03	6.8994E-03	8.5468E+01	5.9953E+03	4 S955E+03
	2.0064E+00	7.4771E-03	6.9744E-03	8.7909E+01	4.6269E+03	3 6085E+03
	2.0159E+00	8.4707E-03	6.7255E-03	8.5903E+01	5.6132E+03	4 4399E+03
31 1017 1542 7 5292E+06 7 2397E+01 32 1017 1605 7 4993E+06 7 2203E+01 33 1017 1615 7 5342E+05 7 2310E+01 34 1017 1622 7 5538E+05 7 2307E+01 35 1017 1625 7 5521E+06 7 2262E+01	2.0038E+00 1.9621E+00 1.9294E+00 1.9093E+00 1.8983E+00	8.4443E-03 8.4286E-03 8.4041E-03 8.3525E-03 8.4415E-03	6.6817E-03 6.6332E-03 6.5388E-03 6.4879E-03 6.5315E-03	8.7202E+01 8.7650E+01 8.8379E+01 8.8791E+01 8.8981E+01	5.5068E+033 5.1655E+033 4.8358E+033 4.6595E+03	4.3057E+03 4.3480E+03 3.9587E+03 3.8293E+03 3.7250E+03
36 1017 1532 7 5428E+06 7 2210E+01 37 1017 1638 7 5198E+06 7 2022E+01 38 1017 1545 7 5476E+06 7 2049E+01 39 1017 1701 7 5502E+06 7 2049E+01 40 1017 1709 7 5309E+06 7 1905E+01	1.8838E+00	8.3772E-03	6.4584E-03	8.9376E+01	4 4131E+03	3.5993E+03
	1.8669E+00	8.5337E-03	6.4644E-03	8.9478E+01	4 4286E+03	3.5083E+03
	1.8476E+00	8.55563E-03	6.4729E-03	8.9919E+01	4 2489E+03	3.4369E+03
	1.8049E+00	8.55642E-03	6.3452E-03	9.1024E+01	3 8231E+03	3.0720E+03
	1.7853E+00	8.5923E-03	6.3566E-03	9.1193E+01	3 7487E+03	2.9986E+03
41 1017 1714 7 5334E+06 7 1963E+01	1.7738E+00	8.5993E-03	6.3413E-03	9.1554E+01	3.6193E+03	2.9186E+03
42 1017 1725 7 5529E+06 7 1906E+01	1.7534E+00	8.6063E-03	6.323BE-03	9.15535E+01	3.42731E+03	2.6989E+03
43 1017 1732 7 5283E+06 7 1770E+01	1.7253E+00	8.5978E-03	6.2304E-03	9.2535E+01	3.1731E+03	2.5226E+03
44 1017 1741 7 5130E+06 7 1485E+01	1.6685E+00	8.5175E-03	6.0764E-03	9.3941E+01	2.5646E+03	2.0020E+03
45 1017 1746 7 5467E+06 7 1523E+01	1.6346E+00	8.5967E-03	6.0267E-03	9.4722E+01	2.2579E+03	1.7454E+03
46 1017 1753 7 5286E+06 7 1531E+01	1.6127E+00	6.9117E-03	5.0096E-03	9.7665E+01	1.4502E+03	1:1272E+03
47 1017 1756 7 5178E+06 7 1287E+01	1.5910E+00	7.4822E-03	5.2755E-03	9.7503E+01	1.4586E+03	1:0987E+03
48 1017 1803 7 5231E+05 7 1275E+01	1.5662E+00	7.2434E-03	5.2755E-03	9.8486E+01	1.2129E+03	8:9521E+02
49 1017 1809 7 5076E+06 7 1187E+01	1.5458E+00	6.4136E-03	4.5497E-03	1.0004E+02	1.2129E+03	6:2138E+02
50 1017 1812 7 4965E+06 7 0874E+01	1.5264E+00	8.2484E-03	5.5311E-03	9.8102E+01	1.0333E+03	6:9775E+02
51 1017 1815 7 5241E+06 7 0916E+01 52 1017 1818 7 5293E+06 7 0943E+01 53 1017 1825 7 5250E+06 7 0719E+01 54 1017 1825 7 5250E+06 7 0533E+01 55 1017 1832 7 4288E+06 7 0256E+01 55 1017 1837 7 3623E+06 7 0051E+01 57 1017 1837 7 3623E+06 7 0051E+01 58 1017 1839 7 3156E+06 6 9872E+01	1.5103E+00 1.4958E+00 1.4640E+00 1.4640E+00 1.4481E+00 1.4413E+00 1.4351E+00 1.4333E+00	8 4575E-0033553 4575E-003553 4575E-003553 45765E-0035 45765E-0035 45765E-0035	3 9033E-03	9 81875+01 9 85775+01 1 9 99195+02 1 022925+02 1 023735+02 1 0436	9 70 18 E + 0 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	6 0239E+02 4 7853E+02 1 78532E+02 5 1307E+01 3 1402E+01 3 0997E+01 2 6644E+01 2 4797E+01

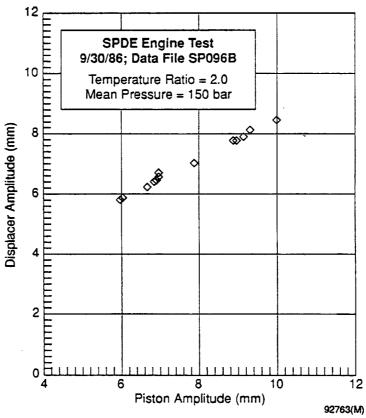
		Data	Summary Rep	ort from:	SP106D::45	at 2:47		OCT., 1986		ile: SPL0iD::45
Nq	Date	Time	PHEAN	FRG(XPL)	TRTOA	XPA 6	XDA ₇	XDPA 8	PVPŞTS	KW(ALT)
100045	4020	1139	7 5244E+06 7 4803E+06 7 4865E+06 7 5023E+06 7 4830E+06	7.1815E+01 7.1739E+01 7.1846E+01 7.2021E+01 7.2113E+01	1.6433E+00 1.6411E+00 1.6987E+00 1.7557E+00 1.8137E+00	5.4472E-03 5.2429E-03 5.4529E-03 5.3828E-03 5.4130E-03	4 1328E-03 3 9682E-03 4 2504E-03 4 2835E-03 4 3764E-03	9.7701E+01 9.8208E+01 9.6219E+01 9.4859E+01 9.3799E+01	1.1736E+03 1.0624E+03 1.4249E+03 1.5999E+03 1.8832E+03	8.8373E+02 8.1923E+02 1.1114E+03 1.2818E+03 1.4708E+03
6 7 8 9 10	1020 1020 1020 1020 1020	1232 1244 1304 1306 1341	7 4866E+06 7 5187E+06 7 5006E+06 7 5057E+06 1 0022E+07	7.2478E+01 7.2651E+01 7.2567E+01 7.2584E+01 8.3186E+01	1.9445E+00 2.0066E+00 2.0007E+00 1.9983E+00 2.0081E+00	5.3794E-03 5.4103E-03 5.3792E-03 5.3733E-03 5.3738E-03	4.5236E-03 4.5972E-03 4.5978E-03 4.55682E-03 4.8756E-03	9.0829E+01 8.9900E+01 8.9737E+01 9.0032E+01 9.1280E+01	2.3486E+03 2.5892E+03 2.5892E+03 2.5830E+03 4.2013E+03	1.8404E+03 2.0220E+03 1.9859E+03 1.9774E+03 2.9468E+03
112334	1020 1020 1020	1345 1347 1355	1.0014E+07 1.0024E+07 1.0028E+07 1.0033E+07 1.0028E+07	8.3182E+01 8.3198E+01 8.2988E+01 8.2991E+01 8.2992E+01	2.0060E+00 2.0061E+00 1.9917E+00 2.0042E+00 2.0058E+00	5.3718E-03 5.3624E-03 6.5037E-03 6.3778E-03 6.3807E-03	4 : 8552E-03 4 : 8461E-03 5 : 6504E-03 5 : 5643E-03	9.1672E+01 9.1821E+01 9.1033E+01 9.1437E+01 9.1000E+01	4 1886E+03 4 2009E+03 5 7128E+03 5 4468E+03	2.9440E+03 2.9240E+03 4.0050E+03 3.9013E+03 3.9133E+03
16 17 18 19 20	1020	1 472	1 2547E+07 1 2539E+07 1 2551E+07 1 25529E+07 1 2525E+07	9.2316E+01 9.2309E+01 9.2322E+01 9.2062E+01 9.2037E+01	2.0010E+00 2.0004E+00 1.9979E+00 2.0080E+00 2.0037E+00	5.3351E-03 5.3400E-03 5.3414E-03 6.3861E-03 6.3568E-03	5.0246E-03 5.0574E-03 5.0574E-03 5.7546E-03 5.7546E-03	9 3129E+01 9 2840E+01 9 3110E+01 9 2337E+01	5963E+03 5963E+03 5968E+03 5968E+03 74857E+03	3.7224E+03 3.7307E+03 3.7096E+03 5.0874E+03 5.0326E+03
20040	1020 1020 1020 1020 1020	1530 1533 1534	1.2521E+07 1.5011E+07 1.5010E+07 1.5013E+07 1.4997E+07	9.2037E+01 1.0022E+02 1.0022E+02 1.0022E+02 9.9967E+01	2.0067E+00 2.0071E+00 2.0071E+00 2.0032E+00 2.0089E+00	6.3665E-03 5.2611E-03 5.3005E-03 5.2952E-03 6.3206E-03	5.7716E-03 5.1555E-03 5.1955E-03 5.9532E-03	9 2111E+01 9 4134E+01 9 4056E+01 9 3971E+01 9 2717E+01	7 5315E+03 7 18E+03 7 2316E+03 7 2352E+03 9 9046E+03	5.0439E+03 4.3572E+03 4.4335E+03 4.3950E+03 6.1171E+03
33389 3389 300 300 300 300 300 300 300 300 300 30	1020 1020 1020 1020 1020	1559	1.5029E+07 1.5015E+07 1.5017E+07 1.5027E+07 7.53315+06	9 9959E+01 9 9641E+01 9 9751E+01 9 9630E+01 7 2538E+01	2 0057E+00 2 0088E+00 2 0026E+00 2 0133E+00 1 9823E+00	6.3331E-03 7.3936E-03 7.2812E-03 7.3953E-03 7.5345E-03	5.9463E-03 6.6193E-03 6.5616E-03 6.6121E-03 6.1257E-03	9 3014E+01 9 2039E+01 9 1977E+01 9 2084E+01 8 8051E+01	9.8808E+03 1.2376E+04 1.2376E+04 1.2367E+04 1.2367E+03	6.0926E+03 7.7018E+03 7.7098E+03 7.6885E+03 3.5930E+03
31 32 33 35 35 35 35	1020 1020 1020 1020 1020	1747 1801 1822 1836 1900	7.5428E+06 7.5473E+06 7.5185E+06 7.5157E+06 7.5152E+06	7 2467E+01 7 2256E+01 7 2089E+01 7 1876E+01 7 1438E+01	1.9433E+00 1.8920E+00 1.8381E+00 1.8057E+00 1.7143E+00	7.4217E-03 8.5649E-03 8.5173E-03 8.5315E-03 1.0715E-02	6.0087E-03 6.6477E-03 6.5523E-03 6.4424E-03 7.4885E-03	8 8570E+01 8 8376E+01 8 9196E+01 9 0558E+01 8 9532E+01	4 2403E+03 4 9293E+03 4 3786E+03 4 0102E+03 4 4370E+03	3.3218E+03 3.8583E+03 3.5308E+03 3.2536E+03 3.4647E+03
36789740	1020 1020 1020 1020 1020	1917 1951 1955 2000 2006	7.5210E+06 7.5140E+06 7.5311E+06 7.5177E+06 7.5054E+06	7.1261E+01 7.0585E+01 7.0603E+01 7.0618E+01 7.0451E+01	1 6223E+00 1 4632E+00 1 4550E+00 1 4452E+00 1 4514E+00	1.0661E-02 1.0027E-02 8.6005E-03 7.4340E-03 4.0835E-03	7.2928E-03 6.4078E-03 5.5768E-03 4.9021E-03 2.6964E-03	9 1002E+01 9 4335E+01 9 9364E+01 1 0155E+02	2 9990E+03 5 6473E+02 4 2536E+02 2 4015E+02 1 5342E+02	2.2937E+03 3.2711E+01 7.5315E+01 6.1070E+01 3.9570E-01

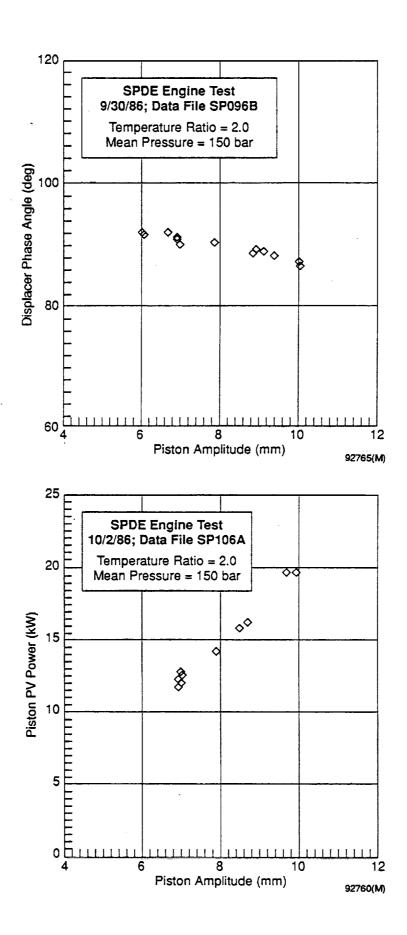
No 1	Date		Summarv Rep PMEAN 3	FRQ(XPL)	SP106E::45 TRTOA	at 4:36 XPA 5	PM FRI.; 24 XDA	OCT., 1986 XDPA	Plot : PVPSTS	file: SPL1E::45 KW(ALT) 10
1234	1024 1024 1024 1024	0957 1012 1049 1118	7.5577E+06 7.5200E+06 7.5307E+06 7.5089E+06 7.5192E+06	7 2006E+01 7 1939E+01 7 2584E+01 7 2577E+01 7 2785E+01	1.5921E+00 1.6155E+00 1.7114E+00 1.8135E+00 1.9125E+00	5.0658E-03 5.0640E-03 6.0158E-03 5.9335E-03	4.0431E-03 4.1060E-03 5.0362E-03 5.1288E-03 5.2934E-03	9.7790E+01 9.6716E+01 9.2216E+01 8.9465E+01 8.7541E+01	8.7719E+02 1.0017E+03 1.9901E+03 2.4598E+03 2.9670E+03	6.7084E+02 7.3468E+02 1.5104E+03 1.9567E+03 2.3179E+03
7 8 9	1024 1024 1024	1200 1215 1217	7.5043E+06 7.4615E+06 1.0020E+07 1.0005E+07 1.2497E+07	7.2903E+01 7.2699E+01 8.3291E+01 8.3272E+01 9.2224E+01	2.0226E+00 2.0241E+00 1.9990E+00 1.9996E+00 2.0200E+00	5.9638E-03 5.9519E-03 5.9875E-03 5.9875E-03 6.0041E-03	5.4325E-03 5.4258E-03 5.6912E-03 5.6874E-03 5.9944E-03	8.5710E+01 8.5732E+01 8.9160E+01 9.9133E+01 8.9771E+01	3.5061E+03 3.4770E+03 5.3946E+03 5.3223E+03 7.8206E+03	2.7152E+03 2.7028E+03 3.9082E+03 3.9267E+03 5.2992E+03
12345	1024 1024 1024 1024 1024	1246 1249 1251 1329	1.2496E+07 1.2518E+07 1.2495E+07 1.5055E+07 1.5051E+07	9.2229E+01 9.2302E+01 9.2245E+01 1.0027E+02 1.0030E+02	2.0197E+00 2.0237E+00 2.0261E+00 2.0183E+00 2.0129E+00	6 0366E-03 5 9778E-03 6 0094E-03 5 9724E-03 6 0300E-03	6.0101E-03 5.8506E-03 5.8665E-03 6.0648E-03 6.1023E-03	9.0049E+01 9.0541E+01 9.0662E+01 9.1473E+01 9.1433E+01	7.9079E+03 7.3781E+03 7.5624E+03 9.9821E+03 9.8763E+03	5.3467E+03 5.1575E+03 5.1575E+03 6.2646E+03 6.2525E+03
18	1024	1350	1.5050E+07 1.5062E+07 1.5023E+07 1.5059E+07 1.5008E+07	1.0031E+02 1.0035E+02 1.0022E+02 1.0027E+02 1.0011E+02	2.0118E+00 2.0093E+00 2.0078E+00 2.0067E+00 2.0050E+00	5 9662E-03 5 9895E-03 5 9807E-03 5 9915E-03 6 0058E-03	6.0109E-03 5.0179E-03 5.9953E-03 6.0178E-03 6.0341E-03	9.1706E+01 9.1598E+01 9.1511E+01 9.1730E+01 9.1580E+01	9.4742E+03 9.4564E+03 9.5800E+03 9.5245E+03 9.4219E+03	6.1440E+03 6.1505E+03 6.0464E+03 6.0385E+03 6.0504E+03
222240	1024 1024 1024 1024 1024	1417 1426 1437 1446 1455	1.5046E+07 1.5055E+07 1.5034E+07 1.5051E+07 1.5057E+07	1.0020E+02 1.0022E+02 1.0023E+02 1.0020E+02 9.7953E+01	2.0050E+00 2.0064E+00 2.0064E+00 2.0068E+00 1.9838E+00	5.0693E-03 6.0073E-03 5.9931E-03 6.0058E-03 7.0331E-03	6.0717E-03 6.0259E-03 6.0084E-03 6.0169E-03 6.7839E-03	9.1538E+01 9.1692E+01 9.1852E+01 9.1792E+01 9.0351E+01	9.7337E+03 9.6442E+03 9.6167E+03 9.5505E+03 1.2064E+04	6.2216E+03 6.1295E+03 6.0260E+03 6.1401E+03 7.8847E+03
200000	4024	1000	1.5100E+07 1.5075E+07 1.5045E+07 1.5091E+07 1.5093E+07	9.9462E+01 9.9543E+01 9.9804E+01 9.9276E+01 9.9322E+01	2 0174E+00 2 0148E+00 2 0441E+00 1 9870E+00 1 9823E+00	9.6437E-03 9.2484E-03 7.9152E-03 1.0375E-02 1.0264E-02	9.4714E-03 8.2080E-03 7.3002E-03 8.9554E-03 8.9154E-03	8.7056E+01 8.7640E+01 8.9684E+01 8.5898E+01 8.5971E+01	2.0531E+04 1.9104E+04 1.5371E+04 2.2022E+04 2.1453E+04	1.2976E+04 1.2218E+04 9.8513E+03 1.4414E+04 1.4260E+04
3123 323 334 353	1024 1024 1024 1024 1024	1550 1552 1557 1600 1602	1.5012E+07 1.5047E+07 1.5027E+07 1.5057E+07 1.5028E+07	9.9592E+01 9.9558E+01 9.8947E+01 9.9220E+01 9.9187E+01	2.0650E+00 2.0643E+00 2.0295E+00 2.0210E+00 2.0084E+00	9 3114E-03 8 5599E-03 8 9037E-03 9 8427E-03 9 9199E-03	7.6140E-03 7.7757E-03 7.8710E-03 8.6203E-03 8.6854E-03	8 .9464E+01 8 .9223E+01 8 .9924E+01 8 .7160E+01 8 .6817E+01	1 6830E+04 1 7607E+04 1 8233E+04 2 0726E+04 2 0480E+04	1.0796E+04 1.1385E+04 1.3007E+04 1.3843E+04 1.3851E+04
36 37 38 39 40	1024 1024 1024 1024 1024	1613 1616 1621 1629 1631	1.5056E+07 1.5080E+07 1.5133E+07 1.5099E+07 1.5024E+07	9.9150E+01 9.9150E+01 9.9154E+01 9.9137E+01 9.9372E+01	1.9934E+00 1.9902E+00 1.9830E+00 1.9674E+00 2.0470E+00	1.0622E-02 1.1059E-02 1.1277E-02 1.1631E-02 9.4902E-03	9.1529E-03 9.5155E-03 9.5753E-03 9.7576E-03	8 5266E+01 8 4476E+01 8 4150E+01 8 4080E+01 8 7370E+01	2.2455E+04 2.4000E+04 2.4895E+04 2.4514E+04 1.8703E+04	1.5377E+04 1.6035E+04 1.6783E+04 1.6803E+04 1.3408E+04

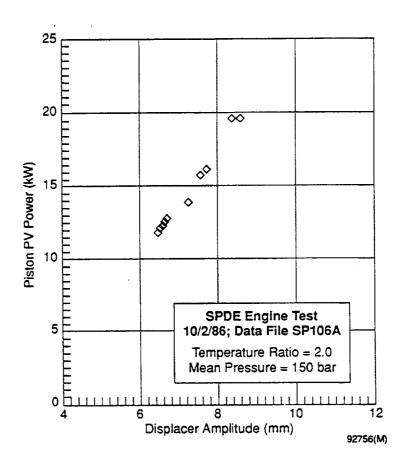
APPENDIX C

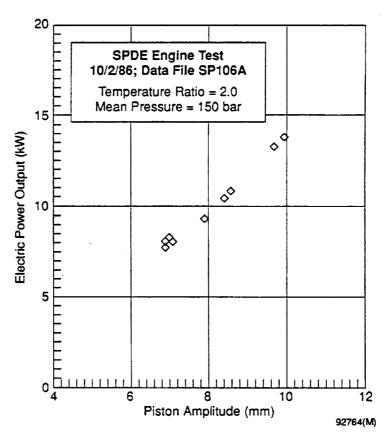
SELECTED SPDE PLOTS PRODUCED FROM APPENDIX B DATA

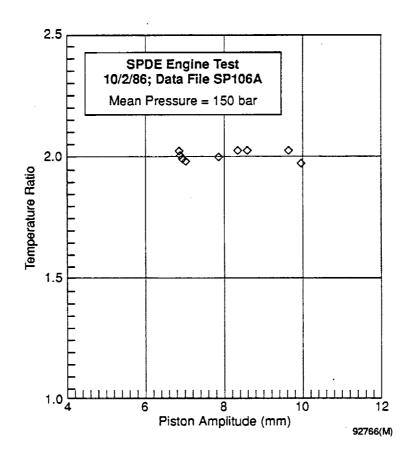


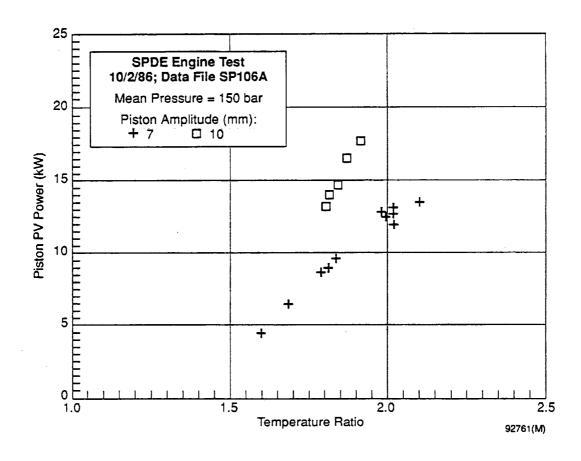


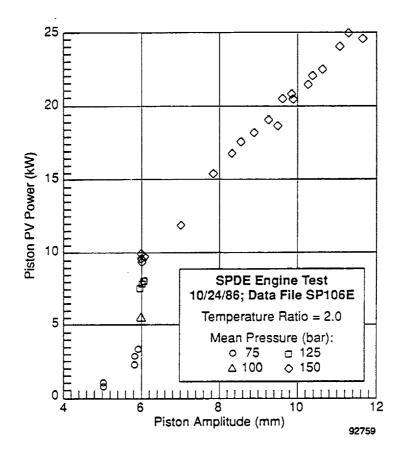


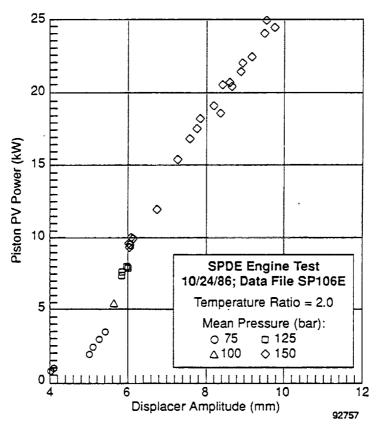


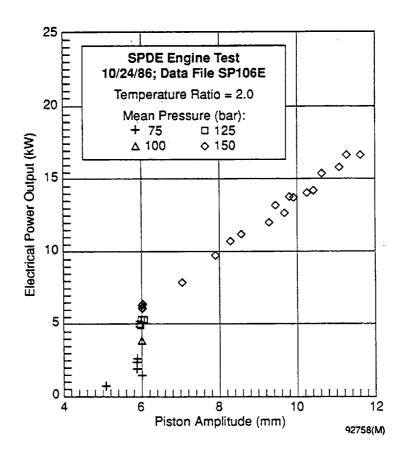












APPENDIX D

SPRE HIGH-EFFICIENCY ALTERNATOR TEST

- Run Sheet
- Inspection and Build Summary
- High-Efficiency Alternator Test Data Plots
- Data Summary Reports for High-Efficiency Alternator Tests

NOMENCLATURE

ACASE Case acceleration (m/sec²)

ACCXC(A) Case acceleration amplitude (m/sec²)

AKADS Aft displacer spring stiffness (N/m)

AKAPS Piston spring stiffness (N/m)

AKFDS Forward displacer spring stiffness (N/m)

AMP First harmonic amplitude

CADS Aft displacer spring damping coefficient (N-s/m)

CAPS Piston spring damping coefficient (N-s/m)

CFDS Forward displacer spring damping coefficient (N-s/m)

DPA Calculated heat exchanger ΔP amplitude (Pa)

DPBNGD Displacer bearing ΔP (Pa)
DPBNGP Piston bearing ΔP (Pa)

DPPH Calculated heat exchanger ΔP phase (deg)
DSFRG Design frequency at operating pressure (Hz)

DTAFFH Average heater fluid film ΔT (°C)
DTBABS Cooler thermocouple ΔT (backup)

DTBAC Alternator cooler thermocouple ΔT (backup) (°C)

DTBEC Engine cooler thermocouple ΔT (backup)

DTBEH Engine heater thermocouple ΔT (backup) (°C)

DTPABC Engine cooler delta temperature (°C)

DTPAC Alternator cooler ΔT (°C)
DTPECn Engine cooler ΔT (°C)
DTPEHn Engine heater ΔT (°C)

DTPLD Load thermocouple ΔT (backup) (°C)

DTSLT Salt heater temperature rise (°C)

ETALT Alternator efficiency

ETCRNO Carnot efficiency (average wall temperature)

ETPVC PV efficiency (based on heat reject)
ETPVP PV efficiency (based on heat input)

ETSYS System efficiency (power output/heat input)

FBNGD Displacer bearing flow ΔP (in. H_2O)

FBNGR Piston bearing flow ΔP (in. H_2O)

FLAC Alternator coolant flow (Vsec)

FLEC Engine coolant flow (Vsec)

FLEH Engine heater flow (Vsec)

FRQDVM Not used

FRQ(SVM) Engine frequency (Hz)

IALT Alternator current (A rms)

IALTD Alternator current (A)

IDC dc load current (A)

KWALT Alternator output power (W)

KWALTM Alternator power output (W)
KW(HTRS) Power to salt heaters (W)

Mean Time averaged value

PADSD Displacer aft gas spring pressure (Pa)

PALTS Alternator shaft power (W)

PAPSP Piston aft gas spring pressure (Pa)

PBRNGD Displacer bearing supply pressure (Pa)

PBRNGP Piston bearing supply pressure (Pa)

PCA Compression space pressure amplitude (Pa)

PCL Compression space pressure (Pa)

PCPH Compression space pressure phase with respect to XP (deg)

PCPHI Ideal pressure phase (deg)

PCPM Compression space/mean pressure amplitude ratio
PEA Calculated expansion space pressure amplitude (Pa)

PEPH Calculated expansion space pressure phase (deg)

PES Expansion space pressure (Pa)

PFDSD Displacer forward gas spring pressure (Pa)

PHADS Aft displacer spring phase (deg)

pressure phase (deg)
espect to piston amplitude
espect to displacer
lacer spring pressure phase (deg)
re (Pa)
oft displacer spring power (W)
pring power (W)
forward displacer spring power (W)
inimum pressure ratio
sure (bar)
piston PV power (W)
ower (W)
spring power (W)
power (W)
er to casing (W)
placer spring power (W)
rnator cooler heat reject (W)
eat reject (backup) (W)
eat reject (prime) (W)
ine cooler heat reject (W)
reject (backup) (W)
reject (prime) (W)
ine heater heat input (W)
input (backup) (W)
input (prime) (W)
om)

TAECW	Cooler wall temperature average (°C)
TAEHW	Average heater wall temperature (°C)
TALTL	Left alternator stator temperature (°C)
TAMBI	Inside ambient temperature (°C)
TAMBO	Outside ambient temperature (°C)
TBABSI	Cooler inlet temperature (backup) (°C)
TBACI	Alternator inlet temperature (backup) (°C)
TBECI	Engine cooler inlet temperature (backup) (°C)
TBECW	Cooler wall temperature (backup) (°C)
TBEHI	Engine heater inlet temperature (backup) (°C)
TBEHW	Heater wall temperature (backup) (°C)
TBRGR	Piston bearing return thermocouple temperature (°C)
TBRGSD	Displacer bearing supply thermocouple temperature (°C)
TBRGSP	Piston bearing supply thermocouple temperature (°C)
TCCR3	Cold regenerator 3:00 o'clock temperature (°C)
TCCR9	Cold regenerator 9:00 o'clock temperature (°C)
TCCRL6	Cold regenerator 6:00 o'clock temperature (°C)
TCCRL12	Cold regenerator 12:00 o'clock temperature (°C)
TCEXP1	Expansion space thermocouple temperature (°C)
TCHRL1	Hot regenerator thermocouple temperature (°C)
TCHRL2	Hot regenerator thermocouple temperature (°C)
TCSL1	Left compression space thermocouple temperature (°C)
TCSRF1	Expansion space thermocouple temperature (°C)
TCSRF2	Expansion space thermocouple temperature (°C)
TFDGSL	Forward displacer gas spring left thermocouple temperature (°C)
thtrent1	Salt heater control temperature (°C)
THTRI	Salt heater inlet temperature (°C)
THTRO	Salt heater outlet temperature (°C)
TPABCI	Engine cooler inlet temperature (°C)
TPACI	Alternator cooler inlet temperature (prime) (°C)
TPCYLL	Left piston cylinder temperature (°C)
TPECIn	Engine cooler inlet temperature (prime) (°C)
TPECW	Cooler wall temperature (prime)

TPEHIn	Engine heater inlet temperature (prime) (°C)
TPEHW	Heater wall temperature (prime) (°C)
TPLDI	Load inlet temperature (prime) (°C)
TPPD1	Displacer position probe temperature (°C)
TREF-1	Thermocouple reference temperature (°C)
TREF-2	Thermocouple reference temperature (°C)
TREF-3	Thermocouple reference temperature (°C)
TREF-8	Reference suction temperature (°C)
TRTEC	Expansion/compression temperature ratio
TRTOA	Heater/cooler wall temperature ratio (average)
TRTOB	Heater/cooler wall temperature ratio (backup)
TRTOFB	Heater/cooler fluid temperature ratio (backup)
TRTOFP	Heater/cooler fluid temperature ratio (prime)
TRTOP	Heater/cooler wall temperature ratio (prime)
TRTRG	Regenerator temperature ratio
TSPOL1	Spool temperature engine end (°C)
TSPOL2	Spool temperature mass end (°C)
VACLD	Alternator load voltage (V rms)
VALT	Alternator terminal voltage (V)
VALTL	Alternator terminal voltage (V rms)
VCAP	Series capacitor voltage (V rms)
VCAPD	Tuning capacitor voltage (V)
VDC	dc load voltage (V)
VLD	Alternator load voltage (V)
VPA	Piston velocity amplitude (m/sec)
VSERLD	Series load voltage (V rms)
XCA	Calculated casing amplitude (m)
XCPH	Calculated casing phase with respect to XP (deg)
XDA	Displacer amplitude (m)
XDL	Displacer displacement (m)
XDL1	Mean displacer position (m)

XDL2	Displacer amplitude (m)
XDL(A)	Displacer amplitude (m)
XDLCK	Displacer displacement check (m)
XDL(M)	Mean displacer position (m)
XDMA	Mean displacer position (m)
XDPH	Displacer phase with respect to piston (deg)
XDRP	Displacer/piston amplitude ratio
XDSPET	$XDA \times SIN(XDPH) \times ETCRNO (m)$
XPA	Piston amplitude (m)
XPL	Piston displacement (m)
XPL1	Mean piston position
XPL2	Piston amplitude (m)
XPL(A)	Piston amplitude (m)
XPL(M)	Mean piston position (m)
XPMA	Mean piston position (m)

Test Configuration Summary

This test is the second test for the Hi-Efficiency Alternator evaluation. The engine configuration used standard hydrostatic bearings, 1.0 mil Brunswick regenerator material, spacer ring removed from pressure vessel.

This test was conducted after the flange/post to cylinder pins were redrilled to correct a misalignment problem discovered on the last test.

The Alternator efficiency was very close to code predictions, a major improvement from the magnetic material tested for the baseline test. Also the PV power and subsequently Alternator Power were hiogher than previously attained.

Test Engineer Date

S P R E ENGINE RUN SHEET

OPERATOR: CWalf	DATE: 01/03/90
RUN NO.: <u>50</u> BUILD NO: <u>28</u>	,
SALT PUMP START: 08 : 00 01/03/90;	ENGINE START: <u>09</u> : <u>54</u> <u>01/03/90</u>
STOP: 17: 18 01/03/90;	STOP: 17 : 12 01/03/90
DAY TOTAL HRS: 9.3	DAY TOTAL HRS: 7.5
accum. total hours: 1680.5	ACCUM. TOTAL HOURS: 2443
	BOOST PUMP START: 09 : 5 01/03/9
	STOP: <u>17</u> : <u>12</u> <u>01/05/90</u>
	DAY TOTAL HRS: 7.3
	ACCUM. TOTAL HOURS: 167. O Disales
	166.0 Poston
TEST OBJECTIVES:	`

Bose line map-high eff. alt. test.

COMMENTS/PROBLEMS:

SPRE INSPECTION AND BUILD SU	BUMMARY					i di	PAGE 1	
ENGINE #: 2 BUILD #: 28	BUILD START: 12/ BUILD COMPLETE:		8/89 12/18/89	ENGINEER: TECHNIC	ER: R.Bolton NICIAN:C.Wol	GINEER: R. Bolton TECHNICIAN:C. Wolfe/W. Smith	th	
COMPONENT	P/N 1015	S/N	DESIGN	ACTUAL	WEIGHT	DATE	TECK	COMMENTS
1. HEATER (1632 tubes) 2. DISP. CYL. SEAL	C-0220-F D-0060-C	01	(1631 ID 4.5040	tubes)	26.68000	02/18/88 05/20/85	CFW GDA	
3. REGENERATOR Stand off wire 8 ea .032						07/31/89	WJS	
Corase screen 1 ea .030 Brunswick 1 ea .001	B-0234-B C-0218-B				.06680	07/31/89	WJS WJS	
en 4 ea .0	B-0233-B				.01600	07/31/89	WJS	
1 ea .2	C-0218-B				.02400	07/31/89	WJS	
Fille Screen 4 ed .010 Brunswick 1 ea .289	G-0218-B				.23600	07/31/89 $07/31/89$	S CA	
•	B-0234-B				.06680	07/31/89	WJS	
ם מם						0//31/89	2	
4. COOLER (1584 tubes) 5. OUTER VENT ORIFICE 6. INHER VENT ORIFICE 7. NUTS (24)	D-0068-E B-0147-B B-0147-B	01	(1584 ID 0.006 ID 0.006	tubes) 00.0060 00.0060	9.26500 20.00200 20.00200 30000	02/16/88 02/16/88 02/16/88 04/01/88	CFW	
9. COOLER (1/O FLANGES (2))9. BOLTS (8)	C-0130-B	162			.69400	05/24/85 05/24/85	JSR	
10. PRESSURE VESSEL W/STUDS 11. NUTS (30)	D-0501-A	01			19.09090 .51970	11/15/89 02/19/88	CFW	
12. ALT. COOLER JACKET	C-0123-C	01			5.85500	03/31/88	CFW	
14. SPACERS (4)	B-0301-A	1-4			03000	03/28/88	CFW	
15. DISP. DOME ASSEMBLY *1 16. FORWARD G.S. SEAL 17. DISPLACER EXP/CMP SEAL	C-0037-	01	ID 3.3514 OD 4.5000	1 3.3514 3 4.50015		$\begin{array}{c c} 08/10/89\\ 08&10/89 \end{array}$	SRI SRI	

Note: All length units are in inches.

SPRE INSPECTION AND BUILD BU	ВОММАВУ					74	PAGE 2	
ENGINE #: 2 B	BUILD START: 12, BUILD COMPLETE:	: 12/08, ETE: 12,	/08/89 12/18/89	ENGINEER: TECHNICIA	ER: R.Bolton CIAN: C.Wolfe	ENGINEER: R. Bolton TECHNICIAN: C. Wolfe/W. Smith		
COMPONENT	P/N 1015	s/N	DESIGN	ACTUAL	WEIGHT Kg	DATE	TECH	COMMENTS
18. DISPLACER ROD *1 19. BOLTS & WASHERS (4) *1 20. DISP. ROD BRG./SEAL	D-0070-H	1	OD 1.8000	1.8000	.02954	$\frac{09/27/89}{08/18/89}$	CFW	
21. GAS SPRING PISTON *1 22. BOLT'S & WASHERS (4) *1 23. GAS SPRING PISTON SEAL	D-0595-A	01-P3	OD 3.2640	3.2648	.25764	09/27/89 09/27/89 09/05/89	CFW CFW TB	
24. F/P W/INSTRUMENTATION 25. BORE/BEARING SEAL 26. G.S. SEAL 27. FIXTURE BOLTS (4)	D-0113-D	03	ID 1.8010 OD 3.3500	1.8010	9.31600	02/18/88 10/19/87 10/19/87 10/19/87	CFW DNS DNS DNS	
28. GAS SPRING CYLINDER 29. BOLTS (8) 30. AFT G.S. SEAL 31. STUFFER VOLUME	D-0106-C	01-P3	ID 3.2659 (6.3060	13.2666 in**3)	3.70300	10/19/87 10/19/87 10/19/87 05/27/87	DNS DNS DNS JSR	
32. JOIN'N RING W/INSTESTUDS	D-0504-A	01			29.91477	11/11/89	CFW	
33. PISTON CYLINDER W/PLUGS 34. INNER STATOR & NUTS 35. BOLTS (9) 36. G.S. BRG. SUPPLY PORT 37. BRG. RET. ORIFICE 38. CYLINDER BORE	D-0502-A D-0488-A PLUG	01	X OPEN ID 0.0200 ID 5.7000	CLSD 5.7014	.03000	04/25/85 03/20/89 04/01/88	JSR CFW CFW	
39. POWER PISTON W/STUDS *2 40. PISTON BRG. SEAL	D-0088-C	01	06 Ports 0D 5.7000	Open 5.7003		12/16/87	CFW	
41. PLENUM COV/ARM MNT *2 42. BOLTS (18)	D-0276-D	01			1.26200	03/30/88 03/30/88	CFW	

Note: All length units are in inches.

HAPECTION AND BUILD	винмаку	1				đ	PAGE 3	
ENGINE 1: 2 BUILD 1: 28	BUILD START: 12, BUILD COMPLETE:	12/ TE:	12/08/89 E: 12/18/89	ENGINEER: TECHNICIA	ENGINEER: R.Bolton TECHNICIAN:C.Wolfe/W.Smith	on fe/W.Smith		
COMPONENT	P/N 1015	S/N	DESIGN	ACTUAL	WEIGHT	DATE	TECH	COMMENTS
43. STATOR MTG. RING 44. BOLTS (12)	E-0277-E	01			Kg 2.43230 .03500	03/30/88 03/30/88	CFW	
45. ALTERNATOR PLUNGER *2 46. NUTS (18) *2 47. MAGNET DIAMETER 48. MAGNET DIAMETER	D-0036-C	05	ID 8.367 OD 8.940	8.3600 8.9700	4.18950	02/19/88 02/19/88 02/19/88 02/19/88	CFW CFW CFW	
49. OUTER ALTERNATOR STATOR W/STUDS & NUTS 50. STATOR ID	D-0284-B	02	ID 9.000	0000.6 0000.6	26.60000	02/19/88	CFW	
BEARING CLEARANCES 51. DISPLACER ROD 52. POWER PISTON			0.0010	10 0.0010 10 0.0011	10 11		: ;	
SEAL CLEARANCES 51. DISPLACER EXP/CMP (2,18) 54. FWD DISPLACER G.S. DISPLACE 55. FWD DISPLACER G.S. PISTON (56. AFT DISPLACER G.S. PISTON (57. AFT DISPLACER G.S. ROD (20,58. PISTON CMP. SPACE (38,40) 59. PISTON GAS SPRING (38,40)	(2,18) DISPLACER (16,26) PISTON (20,25) PISTON (23,30) ROD (20,25) (38,40) (38,40)	G	0.0040 0.0014 0.0010 0.0014 0.0010	40 0.0041 14 0.0012 10 0.0010 14 0.0016 10 0.0011 10 0.0011	141 110 110 111			
ALTERNATOR PLUNGER CLEARANCES 60. INNER GAP 61. OUTER GAP	ES		090.0	0 0.0600	00			

Note: All length units are in inches.

PAGE 4	.Smith	E TECH COMMENTS	
	lton lfe/	DATE	
	ENGINEER: R. Bolton TECHNICIAN:C.Wolfe/W.Smith	WEIGHT	
	ENGINE	ACTUAL WEIGHT	
	BUILD START: 12/08/89 BUILD COMPLETE: 12/18/89	DESIGN	
	: 12/C :TE: 1	s/N	
	START:	1015	SS
BUMMARY	BUILD 6	P/N 1015 S/N	IC MASS NAMIC MASS IC MASS
BUILD	11		DYNAM BLY DY DYNAM SS
BPRE INSPECTION AND BUILD BUMMARY	SUGINE #: 2 BUILD #: 28	COMPONENT	TOTAL DYNAMIC MASS 62. PISTON ASSEMBLY DYNAMIC 63. DISPLACER ASSEMBLY DYNAM 64. CASING ASSEMBLY DYNAM 65. TOTAL ENGINE MASS
BPRE	ENGI	COME	TOTA 62. 63. 64.

NOTES:

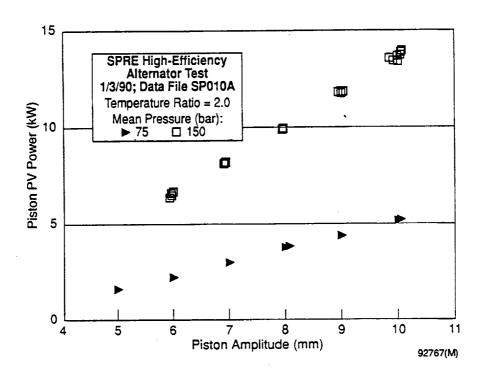
DISPLACER MASS COMPONENTS PISTON MASS COMPONENTS

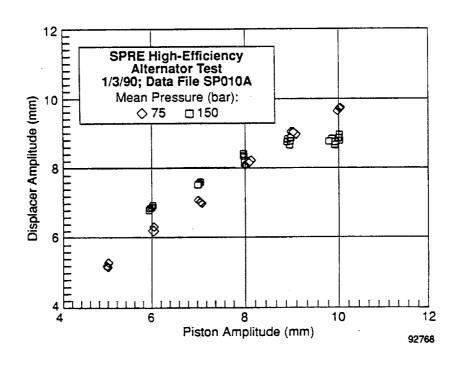
DISPLACER OUTSTOP MODIFICATION (ref. build 12) MOD. HEIGHT FROM.220" TO.160"
PISTON OUTSTOP.075"
DISPLACER CENTERING INSTALLED
ADDED STAND OFF WIRE TO BOTTOM OF COOLER 0.032 in.
COMPRESSION SPACE TRANCDUCER EXT. TUBE ADDED 5.25"X.126"X.082"
EXPANSION SPACE PRESS. XDUCER REPLACED BY FOUR TC's -2 on disp. cyl.(top/bot), 2 on post(top/bot)
REMOVED TC CYL(bot), REPLACED WITH COMP.SPACE TC 3615
ONE TC (3708) TO DISPLACER AFT GAS SPRING
TACKED FOWARD DISPLACER GAS SPRING TC (3703) TO THE SEAL I.D. WALL. CHANGED TO FWD.DISP. SEAL WALL ADDED 8 TC'S TO FLANGE/POST ASSEMBLY; I.D'd 0 & 7-14, READ ON FLUKE 2280B DATA LOGGING SYSTEM. FOR LOCATION, SEE ATTACHED ENCLOSURES.

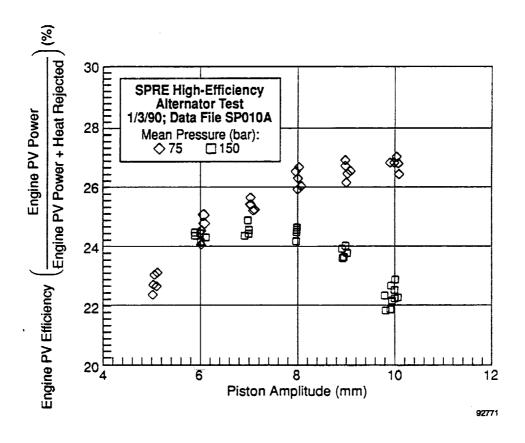
REMOVED TWO TC'S ON DISPLACER CYLINDER 3710 & 3712 DISPLACER VENT HOLE MOD. TO 1/4-20 TAP, 13.5 MIL HOLE IN 1/4-20 CAPSCREW W/CU. GASKET AS VENT. NEW WASHER DESIGN UNDER 4-10-32 X 1 3/4" ROD TO DISPLACER CAPSCREW.

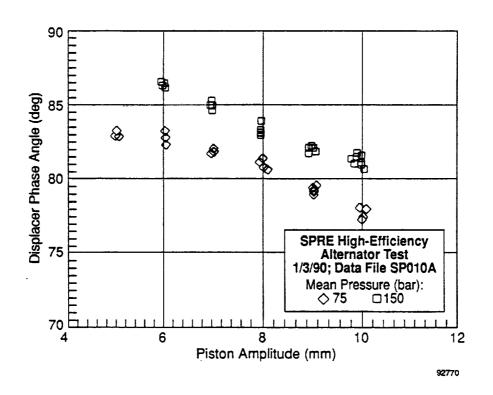
ONE TIME NOTE; DISPLACER CYLINDER/FLANGE-POST REPINNED. MIN 1mil CLEARANCE ESTABLISHED

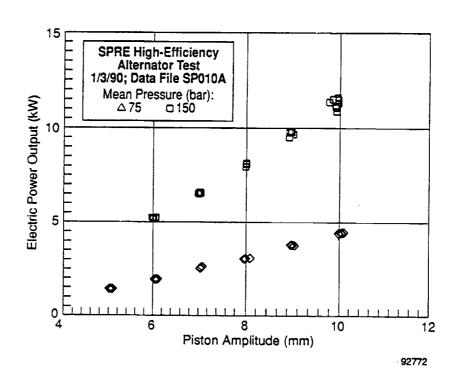
Note: All length units are in inches.

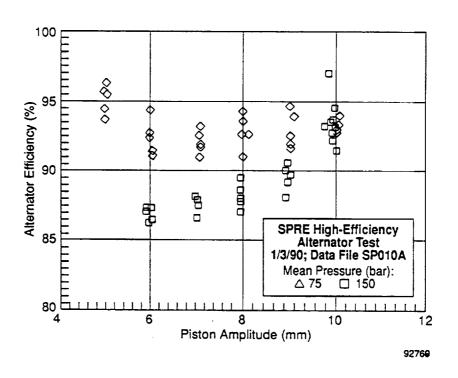


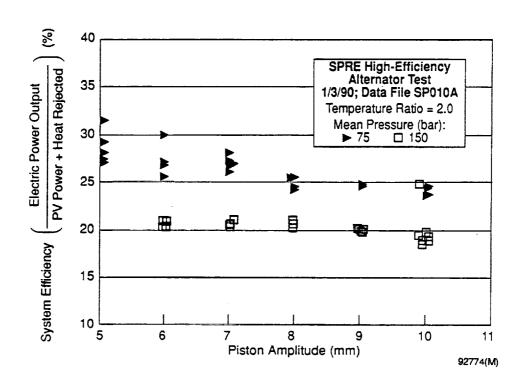












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CELL 5 - REGENERATOR TEST - CHMSkin (8912:23 1525) MEASUREHENT SCAN. 67 1-3 1990 15.23:37 (FILE. SP010

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,	-1,3320E-03	9.9804E-03	6,3384E+00		8.8451E-03	8,1156E+01
9 6	9.9924E+01	4.5082E-05	-1,7169E+02		1.3593E-01	-7. 3302E+00
ì	1 2694E+03	1,6222E+03	9 1433E+05		1.8320E+02	-1.5065E+01
	2.7831E+02	3.5565E+02	1 0732E+05		1.85%E+82	1.3606E+04
, ,	1,2807E+03	1,2857E+03	1 1960E+06		1,8194E+02	1.3146E+00
	5 9789E+04	4.8354E+04	1 6641E+03		-1.5533E+01	1.9129E-01
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>	1.9536E+01	S.3309E+02	3.8874E-03	0.0000E+00	0.0000E+00	

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Mechanical Technology Inc	corporated (MTI) performed acce	eptance testing on the Space Po	wer Research Engine					
(SPRE), which demonstrate	ed satisfactory operation and suff 13.5 kW PV power with an effici	icient reliability for delivery to	ols of 28 8 kW PV nower					
	cimum electric power was only 8							
	SPRE was linear alternator effici							
of 90%. It was determined	from static tests that the major ca	ause for the efficiency shortfall	was the location of the					
magnetic structure surround	ding the linear alternator. Testing	of an alternator configuration	without a surrounding					
magnetic structure on a line	ear dynamometer confirmed earli r 90%. Testing of the MTI SPRE	ier static test results. Linear alte	ernator efficiency					
achieved full-stroke stable	operation. This testing indicated	, was also performed with flydd I that hydrodynamic hearings m	odynamic bearings and lay be useful in free-					
piston Stirling engines. An	important factor in achieving sta	ble operation at design stroke v	was isolating a portion of					
piston Stirling engines. An important factor in achieving stable operation at design stroke was isolating a portion of the bearing length from the engine pressure variations. In addition, the heat pipe heater head design indicates that								
integration of a Stirling engine with a heat source can be performed via heat pipes. This design will provide a								
baseline against which alter	mative designs can be measured.	,						
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